

EFFECT OF TUBE INCLINATION ANGLE  
ON THE THERMAL AND FLUID DYNAMIC  
PERFORMANCE OF FLAT TUBE HEAT  
EXCHANGER

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## SUPERVISOR'S DECLARATION

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I hereby declare that the work in this thesis is based on my original work except for quotations and citation which have been duly acknowledged. I also declare that it has not been previously or concurrently submitted for any other degree at Universiti Malaysia Pahang or any other institutions.

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## TABLE OF CONTENTS

<b>DECLARATION</b>	
<b>TITLE PAGE</b>	
<b>ACKNOWLEDGEMENTS</b>	<b>ii</b>
<b>ABSTRACT</b>	<b>iii</b>
<b>ABSTRAK</b>	<b>iv</b>
<b>TABLE OF CONTENTS</b>	<b>v</b>
<b>LIST OF TABLES</b>	<b>ix</b>
<b>LIST OF FIGURES</b>	<b>x</b>
<b>LIST OF SYMBOLS</b>	<b>xiii</b>
<b>LIST OF ABBREVIATIONS</b>	<b>xiv</b>
<b>CHAPTER 1 INTRODUCTION</b>	<b>1</b>
1.1 Research Background	1
1.2 Research Problem Statement	4
1.3 Research Objectives	5
1.4 Research Scope	5
1.5 Research Questions	6
1.6 Research Approach	6
1.7 Research Contribution	7
<b>CHAPTER 2 LITERATURE REVIEW</b>	<b>8</b>
2.1 Preamble	8
2.2 Overview of the Heat Exchanger	9

2.3	Classification of Heat Exchangers	9
2.4	Heat Transfer Characteristics	11
2.4.1	Conduction Heat Transfer	11
2.4.2	Convection Heat Transfer	11
2.5	Dimensionless Numbers	12
2.5.1	Reynolds Number	12
2.5.2	Nusselt Number	12
2.5.3	Prandtl Number	13
2.6	Investigative Methods from Previous Studies	13
2.7	Experimental Studies	14
2.8	Numerical Studies	20
2.9	Heat Transfer Enhancing Methods	22
2.9.1	Vortex Generators	22
2.9.2	Extended Surfaces	30
2.9.3	Dimple Surface	35
2.10	Geometrical Factors That Affect Flow and Heat Transfer	38
2.10.1	Effects of Tube Parameters	38
2.10.2	Effects of Fin Parameters	44
2.11	Summary	48
<b>CHAPTER 3 METHODOLOGY</b>		<b>49</b>
3.1	Overview of the Research Methodology	49
3.2	Experimental Work	51
3.2.1	Material Preparation	51
3.2.2	Experimental Setup	53
3.2.3	Experimental Procedure	57

3.2.4	Data Collection	58
3.2.5	Data Reduction	60
3.3	Numerical Analysis	62
3.3.1	Description of the Computational Domain	64
3.3.2	Grid Generation	66
3.3.3	Selection of the Thermo-Physical Properties	67
3.3.4	Governing Equations	68
3.3.5	Boundary Conditions	69
3.3.6	Control Parameters	70
3.3.7	Mesh Independency Test	72
3.4	Design of Experiment	73
3.4.1	Process Optimization Procedure	73
3.4.2	Response Surface Methodology	74
3.4.3	Validation of the RSM	77
<b>CHAPTER 4 RESULTS AND DISCUSSION</b>		<b>78</b>
4.1	Numerical Model Validation	78
4.2	Air Flow Observations	80
4.3	Turbulent Properties Distributions	83
4.4	Temperature Observation	87
4.5	Temperature Variation at the HE Outlet	90
4.6	Heat Transfer Performance of the HE	92
4.6.1	Effect of Air Inlet Velocity	92
4.6.2	Effect of Tube Inclination Angle	94
4.6.3	Effect of Thermal Boundary Condition	96
4.7	Hydraulic Performance of the HE	96



4.7.1	Effect of Air Inlet velocity	97
4.7.2	Effect of Tube Inclination angle	99
4.8	Effect of Tube Inclination Angle on the Goodness Factor ( $j/f$ )	101
4.9	Statistical Analysis	103
4.9.1	Analysis for $Nu$	104
4.9.2	Analysis for Pressure Drop	106
4.9.3	Optimization Parameters	109
4.9.4	Sensitivity Analysis	111
<b>CHAPTER 5 CONCLUSION AND RECOMMENDATIONS</b>		<b>116</b>
5.1	Conclusions	116
5.2	Recommendations	118
<b>REFERENCES</b>		<b>120</b>
<b>APPENDIX A Effect of Tube Inclination Angle (Numerical Data)</b>		<b>130</b>
<b>APPENDIX B Outlet Temperature (Inline)</b>		<b>131</b>
<b>APPENDIX C Outlet Temperature (Staggered)</b>		<b>132</b>
<b>APPENDIX D Statistical Analysis for <math>Nu</math> and <math>\Delta P</math> (Experimental Data)</b>		<b>133</b>
<b>APPENDIX E Optimization Parameters (Experimental Data)</b>		<b>135</b>
<b>APPENDIX F Sensitivity Analysis (Experimental Data)</b>		<b>137</b>
<b>APPENDIX G DOE Model Validation (Numerical Data)</b>		<b>138</b>
<b>APPENDIX H Simulation procedure</b>		<b>139</b>
<b>APPENDIX I Fin Construction</b>		<b>140</b>
<b>LIST OF PUBLICATIONS</b>		<b>141</b>

## LIST OF TABLES

Table 2.1	Classification of the heat exchangers	10
Table 2.2	Experimental studies related to the thermal hydraulic performance of fin-and-tube HEs	18
Table 2.3	Numerical studies related to the thermal hydraulic performance of fin-and-tube HEs	23
Table 2.4	Summary of effect of vortex generators on the performance of fin-and-tube HEs	29
Table 2.5	Summary of the effect of fin type and dimple surface on heat transfer and pressure drop of fin and tube HEs	37
Table 2.6	Summary of the effect of tube parameters on heat transfer and pressure drop in fin-and-tube HEs	43
Table 2.7	Summary of the effect of fin parameters on the heat transfer and pressure drop in fin-and-tube HEs	47
Table 3.1	Geometric details of the test samples	55
Table 3.2	Comparison between thermocouple reading and thermometer reading	56
Table 3.3	Experimental conditions	57
Table 3.4	Specification of the main data collection components	59
Table 3.5	Thermo-physical properties of the material used in this study	67
Table 3.6	Independent parameters considered in this study and their levels	74
Table 4.1	The geometrical properties of fin-and-tube HE used in the current study	80
Table 4.2	Central composite design arrangement, response and their values for Numerical data of heat transfer and pressure drop of fin-and-flat tube HE	103
Table 4.3	Estimated regression coefficients for Nu for quadratic equation (Numerical data)	104
Table 4.4	ANOVA test for Nusselt number (numerical-data)	104
Table 4.5	Estimated regression coefficients for $\Delta P$ for quadratic equation (Numerical data)	106
Table 4.6	Estimated regression coefficients for $\Delta P$ for quadratic equation after the elimination of non-significant terms (Numerical data)	107
Table 4.7	ANOVA test for $\Delta P$ (numerical-data)	107
Table 4.8	Sensitivities analysis for Nu (in-line, Numerical)	112
Table 4.9	Sensitivities analysis for Nu (staggered, Numerical)	113
Table 4.10	Sensitivities analysis for $\Delta P$ (both configurations, Numerical)	113

## LIST OF FIGURES

Figure 1.1	Tube shapes (a) flat tube (b) circular tube (c) elliptical tube	3
Figure 2.1	The different solution methods of fluid dynamics and their detail description	14
Figure 2.2	Schematic for some of the basic forms of vortex generators	25
Figure 2.3	Image of (a) Plain fin (b) DWVGs in flow-up and (c) DWVGs in flow-down flow directions	27
Figure 2.4	Internal wavy fin and tube (IWFT)	31
Figure 2.5	Louvered fin and its key geometrical parameters	32
Figure 2.6	Various types of spiral fins	34
Figure 2.7	Examples of different fin pitch sizes in fin-and-tube HEs	46
Figure 3.1	Methodology flow chart of the research	50
Figure 3.2	Outside woods for (a) 0°, (b) 30°, (c) 60°, (d) 90°, (e) 120°, (f) 150° for inline configuration	52
Figure 3.3	Fin and flat tube HE with the thermocouples	53
Figure 3.4	Schematic diagram of the experimental apparatus; 1-Inlet, 2-flow straightener, 3- Anemometer, 4- thermometer (testo 110) ,5- Air probe, 6- Temperature controllers, 7- test section, 8- suction fan, 9- frequency inverter, 10- Digital manometer, 11-NI 9219 DAQ, 12-computer, 13- static pressure tip, 14- thermocouples	54
Figure 3.5	Photo of the experimental apparatus with the measuring devices	54
Figure 3.6	Locations of temperature and pressure sensors for in-line arrangement	58
Figure 3.7	Data acquisition process	59
Figure 3.8	Basic steps for ANSYS Fluent	64
Figure 3.9	Schematic diagrams for cross-section of fin-and-tube heat exchanger with (a) in-line and (b) staggered arrangements (all dimensions are in mm)	65
Figure 3.10	Computational domain for (a) in-line and (b) staggered configuration	66
Figure 3.11	Grid distributions for in-line in configuration	67
Figure 3.12	Flow process of pressure-based solution method	71
Figure 3.13	Nusselt number results for zero angle of attack using different mesh numbers at Re= 3500	72
Figure 3.14	Pressure drop results for zero angle of attack using different mesh numbers at Re= 3500	73
Figure 3.15	Schematic diagram for independent parameters	74
Figure 3.16	Response Surface Methodology flow chart	76

Figure 4.1	Comparison of the Numerical results for the present study simulation with the experimental results of Zukauskas (Zukauskas, 1987) correlation and Gholami et al. (Gholami et al., 2014) numerical	79
Figure 4.2	Air velocity distribution for inline at $V_{in} = 2.3\text{m/s}$ (a) $0^\circ$ , (b) $90^\circ$	81
Figure 4.3	Air velocity distribution for staggered at $V_{in} = 2.3\text{m/s}$ (a) $0^\circ$ , (b) $90^\circ$	81
Figure 4.4	Velocity vectors between tube bundle for inline at $V_{in} 2.3 \text{ m/s}$ , (a) $0^\circ$	83
Figure 4.5	Turbulent kinetic contours energy per unit mass for (a) $\alpha=0^\circ$ , (b) $\alpha=60^\circ$ , (c) $\alpha=90^\circ$ , and (d) circular at $V_{in} =1.8 \text{ m/s}$	85
Figure 4.6	Turbulent kinetic energy contour for inline tube configuration at $0^\circ$ and $90^\circ$ , $V_{in} = 2.3\text{m/s}$	86
Figure 4.7	Turbulent kinetic energy contour for staggered tube configuration at $0^\circ$ and $90^\circ$ , $V_{in} = 2.3\text{m/s}$	86
Figure 4.8	Isotherm contour for (a) $0^\circ$ , (b) $60^\circ$ , (c) $90^\circ$ (Inline) at $V_{in} = 2.3\text{m/s}$	88
Figure 4.9	Isotherm contours for (a) $V_{in} = 2.3 \text{ m/s}$ , (b) $V_{in} = 3.8 \text{ m/s}$ and $\alpha = 0^\circ$ (staggered)	89
Figure 4.10	Mean temperature at different locations for different angles (inline-numerically)	90
Figure 4.11	Comparison between average outlet temperature obtained from experimental and numerical for $0^\circ$ , inline	91
Figure 4.12	Variation of average $Nu$ against $Re$ (in-line, $\alpha = 0^\circ$ )	93
Figure 4.13	Variation of average $Nu$ against $Re$ (staggered, $\alpha = 0^\circ$ )	93
Figure 4.14	Variation of average $Nu$ versus $Re$ at different tube inclination angles (inline, experimental)	95
Figure 4.15	Variation of average $Nu$ versus $Re$ at different tube inclination angles (staggered, experimental)	95
Figure 4.16	Effect of thermal boundary condition on $Nu$ number	96
Figure 4.17	variation of air side pressure drop versus $Re$ at different tube inclination angle (inline, numerical)	97
Figure 4.18	Variation of friction factor against $Re$ (in-line , $\alpha = 0^\circ$ )	98
Figure 4.19	Variation of friction factor against $Re$ (staggered, $\alpha = 0^\circ$ )	98
Figure 4.20	Variation of friction factor versus $Re$ at different tube inclination angles (Inline, numerical)	100
Figure 4.21	Variation of friction factor versus $Re$ at different tube inclination angles (staggered, numerical)	100

Figure 4.22	Variation of area goodness factor against air velocity at different tube inclination angles (in-line)	102
Figure 4.23	Variation of area goodness factor against air velocity at different tube inclination angles (staggered)	102
Figure 4.24	Residual plots for $Nu$ (a) normal probability plot, (b) residual versus fitted values, (c) histogram and (d) residuals versus the order of the data (Numerical data)	105
Figure 4.25	Residual plots for $\Delta P$ (a) normal probability plot, (b) residual versus fitted values, (c) histogram and (d) residuals versus the order of the data (Numerical data)	108
Figure 4.26	Response optimization plot for $Nu$ and $\Delta P$	108
Figure 4.27	Variation of (a) $Nu$ and (b) $\Delta P$ against air velocity (m/s) and tube inclination angle (Numerical data)	110
Figure 4.28	Response surface plots for (a) $Nu$ and (b) $\Delta P$ against air velocity (m/s) and tube inclination angle	111
Figure 4.29	Comparison between the experimental and predicted values for $Nu$	114
Figure 4.30	Comparison between the experimental and predicted values for $\Delta P$	115

## LIST OF SYMBOLS

$\alpha$	Tube inclination angle ( $^{\circ}$ ), thermal diffusivity ( $m^2/s$ )
$\beta$	Heat transfer surface density ( $m^2/m^3$ )
$\delta$	Fin thickness (mm)
$\theta$	Angle of attack, fin inclination angle
$\mu$	Dynamic viscosity, (kg/m.s)
$\rho$	Density ( $kg/m^3$ )
$\sigma$	Deviation
$\infty$	Fluid
$A$	Heat transfer surface area ( $m^2$ )
$Al$	Aluminum
$c$	Characteristics
$Cu$	copper
$cond$	Conduction
$conv$	Convection
$elec$	Electrical
$F$	Friction factor
$F_p$	Fin pitch (mm)
$F_h$	Fin height (mm)
$h$	Convection heat transfer coefficient
$in$	Inlet
$J$	Heat transfer factor
$k$	Thermal conductivity of the tube (W/m.K)
$K$	Turbulence kinetic energy( $m^2/s^2W$ )
$L$	Length of the tube (m)
$\dot{m}$	Mass flow rate (kg/s)
$n$	Number of terms in the model
$N$	Number of tube rows
$Nu$	Nusselt number
$out$	Outlet
$P_l$	Longitudinal tube pitch
$Pr$	Prandtl number
$Q$	Heat transfer rate (W)
$rad$	Radiation
$Re$	Reynolds number
$s$	Surface
$t$	Total
$T$	Temperature ( $^{\circ}C$ )
$U$	Overall heat transfer coefficient
$\nu$	Kinematic viscosity ( $m^2/s$ )
$V$	Air velocity (m/s)
$w$	wall
$x$	Predictor
$X$	Thickness (mm)
$y$	response

## LIST OF ABBREVIATIONS

ANOVA	Analysis of Variance
CCD	Central Composite Design
CDWVGs	Curved delta winglet vortex generators
CHEs	Compact heat exchangers
CFD	Computational Fluid Dynamics
CNC	Computer Numerical Control
CPU	Central Processing Unit
CRVGs	Curved rectangular vortex generators
DO	Direct optimization
DOE	Design of Experiment
DWLVGs	Delta Winglet longitudinal Vortex generators
DWRVGs	Delta Winglet Vortex generators
DWVGs	Delta Winglet rectangular Vortex generators
EU	European Union
Exp	Experimental
Fkm	Fakulti Kejuruteraan Mekanikal
FTHEs	Flat tube heat exchangers
HVAC	Heating ventilation and air conditioning
HE	Heat exchanger
HEs	Heat exchangers
IR	Infrared thermography
LMTD	Log Mean Temperature Difference
LVs	Longitudinal Vortex
NCWPs	Novel combined winglet pairs
PEC	Performance Evaluation Criteria
PID	proportional–integral–derivative
PIV	Particle image velocimetry
PRVG	Punched rectangular vortex generators
PTVG	Punched triangular vortex generators
RSM	Response Surface Methodology
RWVG	Rectangular Winglet Vortex generators
Simu	Simulation
UMP	Universiti Malaysia Pahang
VGs	Vortex generators

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## ABSTRACT

At the present time, the performance of finned and flat tube heat exchangers (HEs) has become a very important issue in the thermal industrial sector. Finned-and-flat tube heat exchangers have gained great interest from many researchers due to their role in any thermal engineering system. Some geometrical and process parameters such as fin spacing, tube spacing, tube inclination angle, etc. affects the performance of fin-and flat tube heat exchangers. Significant number of research have been done to study the effect of such geometrical and process parameters. However, the effect of flat tube inclination angle on the thermal-hydraulic performance of fin-and-tube HE is not fully examined. Thus, the aim of this study was to investigate the effect of flat tube inclination angle, air velocity and tube configuration on the thermal and flow characteristics of compact fin-and-flat tube heat exchangers. A series of experimental and numerical investigations were carried out to evaluate the influence of the aforementioned parameters on the thermal-hydraulic performance between the tube bundles. Moreover, Response Surface Methodology (RSM) was used to determine the optimum parameter condition for the fin-and-tube HE. The range of the parameters considered in the study were tube inclination angle from  $0^\circ$  to  $150^\circ$ , inlet air flow velocity from 1.8 to 3.8 m/s and tube configuration (inline and staggered). For the experiments, the wind tunnel available at the Faculty of Mechanical Engineering, UMP was used. The wind tunnel was equipped with a blower of capacity 50 W, flow straightener, test section and measuring sensors. Twelve plate fin and nine flat tube heat exchangers test sections were designed and manufactured at various inclination angles and configurations. Temperature, velocity, and pressure measurements were recorded at various positions in the test section as well as before and after the test section. For the numerical analysis, the CFD commercial software called ANSYS FLUENT-15 was used to solve the Navier-Stoke and energy equations together with proper turbulent equations. The parameters are similar to the experimental investigation. The experiment and numerical analysis used Nusselt number, pressure drop, and area goodness factor to evaluate the thermal-hydraulic performance of fin-and-flat tube heat exchangers. The major findings showed that flat tube inclination angle has significant impact on the heat transfer enhancement. However, the results from both experimental and numerical analysis revealed that increasing the tube inclination angle from  $0^\circ$  to  $90^\circ$  augments the convective heat transfer coefficient. While  $120^\circ$  and  $150^\circ$  provide thermal performance close to  $60^\circ$  and  $30^\circ$ , respectively. The average deviations of Nusselt number between experimental and numerical results were 5.42% and 4.44% for inline and staggered configurations respectively. Moreover, due to the air flow blockage caused by inclining the tube angle, the pressure drop increases dramatically for all cases studied. From all acquired results, the best thermal performance occurred at  $0^\circ$ , while  $90^\circ$  provided the minimum thermal performance. Therefore, inclining the tube is beneficial for enhancing the heat transfer performance but is not beneficial in term of pressure drop as it requires higher pumping power. The developed correlations from the RSM can predict experimental data with average deviation of 2.78% for  $Nu$  for both configurations. Moreover, it can predict the numerical data with average deviation of 0.554% and 0.92% for inline and staggered configuration, respectively. The optimum parameters are found to be with high air velocity and low tube inclination angle which provide maximum heat transfer enhancement and low-pressure drop penalty. Thus, it is recommended for the design of fin-and-flat tube HEs.

## ABSTRAK

Pada masa kini, prestasi penukar haba tiub bersirip dan rata (HES) telah menjadi satu elemen yang amat penting dalam sektor industri haba. Penukar haba bersirip dan tiub rata telah menarik perhatian ramai penyelidik kerana peranan nya dalam sistem kejuruteraan pengurusan haba. Beberapa geometri dan parameter proses seperti jarak sirip, jarak tiub, sudut kecenderungan tiub, dan lain-lain memberi kesan kepada prestasi penukar haba tiub sirip-dan rata. Sejumlah besar penyelidikan telah dijalankan untuk mengkaji kesan geometri dan parameter proses. Namun, kesan sudut kecenderungan tiub rata ke atas prestasi haba-hidraulik sirip dan tiub HE tidak diperiksa sepenuhnya. Oleh itu, tujuan kajian ini adalah untuk mengkaji kesan sudut kecenderungan tiub rata, halaju udara dan konfigurasi tiub kepada ciri-ciri haba dan aliran penukar haba tiub sirip-dan-rata padat. Satu siri kajian eksperimen dan numerikal telah dijalankan untuk menilai pengaruh parameter yang dinyatakan di atas prestasi terma-hidraulik antara bundle tiub. lebih-lebih lagi, Response Surface Methodology (RSM) telah digunakan untuk menentukan keadaan parameter optimum untuk sirip dan tiub HE. Pelbagai parameter dipertimbangkan dalam kajian ini iaitu: sudut kecenderungan tiub dari sudut  $0^\circ$  hingga  $150^\circ$ , halaju masuk aliran udara 1.8-3.8 m/s dan konfigurasi tiub (dalam baris dan berperingkat-peringkat). Untuk eksperimen ini, terowong angin yang terdapat di Fakulti Kejuruteraan Mekanikal, UMP telah digunakan. Terowong angin ini telah dilengkapi dengan blower kapasiti 50 W, pelurus aliran, bahagian ujian dan sensor ukuran. Dua belas plat tiub sirip dan penukar haba rata bahagian ujian telah direka dan dihasilkan di pelbagai sudut kecenderungan dan konfigurasi. Suhu, halaju, dan tekanan ukuran telah direkodkan pada pelbagai kedudukan dalam ruang sebelum ujian dan selepas bahagian ujian. Untuk analisis numerikal, perisian CFD yang dipanggil ANSYS FLUENT-15 telah digunakan untuk menyelesaikan persamaan Navier-Stoke dan persamaan tenaga bersama-sama dengan persamaan bergelora yang betul. Parameter yang digunakan adalah sama dengan penyiasatan eksperimen. Ujikaji dan analisis berangka menggunakan nombor Nusselt, kejatuhan tekanan, dan kawasan faktor kebaikan untuk menilai prestasi haba-hidraulik penukar haba tiub sirip-dan rata. Dapatan utama menunjukkan bahawa rata sudut tiub kecenderungan mempunyai kesan besar kepada peningkatan pemindahan haba. Walau bagaimanapun, keputusan daripada analisis kedua-dua eksperimen dan numerikal mendedahkan bahawa peningkatan sudut kecenderungan tiub dari  $0^\circ$  hingga  $90^\circ$  menambahkan kadar pemindahan haba perolakan. Manakala  $120^\circ$  dan  $150^\circ$  memberikan prestasi haba hampir kepada  $60^\circ$  dan  $30^\circ$ . Purata sisihan nombor Nusselt antara keputusan eksperimen dan yang berangka adalah 5.42% dan 4.44% untuk masing-masing sebaris dan konfigurasi berperingkat. Selain itu, disebabkan oleh penyumbatan aliran udara yang disebabkan oleh kecenderungan sudut tiub, peningkatan kejatuhan tekanan secara mendadak untuk semua kes yang dikaji. Dari keputusan semua yang diperolehi, prestasi terma yang terbaik berlaku pada  $0^\circ$ , manakala  $90^\circ$  disediakan prestasi termal yang minimum. Korelasi yang diperolehi dari RSM boleh meramalkan data eksperimen dengan sisihan maksimum 2.78% untuk Nu untuk kedua-dua konfigurasi. Selain itu, ia boleh meramalkan data berangka dengan sisihan purata 0.54% dan 0.92% yang sebaris dan konfigurasi berperingkat, masing-masing. Parameter optimum yang diperolehi akan menjadi dengan halaju udara lebih tinggi dan sudut kecenderungan tiub rendah yang menyebabkan peningkatan pemindahan haba maksimum dan tekanan rendah drop penalti. Oleh itu, reka bentuk HEs tiub sirip dan tiub rata adalah disyorkan.

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