

## Article

# Experimental and Numerical Study on an Innovative Trapezoidal-Shaped Damper to Improve the Behavior of CBF Braces

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**Abstract:** Among the existing passive energy dampers, I-shaped shear dampers had shown suitable performance in experimental and numerical studies. Although they improve the dissipating energy and ductility of concentrically braced frames (CBFs), they reduce the stiffness and ultimate strength of the system. Three approaches are generally used to overcome the problem, including (a) increasing the thinness of the shear plates, (b) increasing the number of shear plates, and (c) using more dampers in more bays. The mentioned approaches increase construction costs. Accordingly, to overcome this shortcoming, in this paper, an innovative shear damper with a trapezoidal shape is proposed and investigated experimentally and numerically. The results indicated that when using the same material for I-shaped shear dampers and the proposed damper, the proposed damper has greater ultimate strength, elastic stiffness, and dissipating energy capacity. Additionally, the flange plates are more effective in the behavior of the proposed damper than the I-shaped damper. Moreover, required equations were proposed to design the damper.

**Keywords:** passive dampers; ultimate strength; stiffness; yielding; CBF



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## 1. Introduction

Among conventional lateral load-bearing steel systems, such as Moment Resisting Frames (MRFs) and Eccentrically Braced Frames (EBFs), Concentrically Braced Frames (CBFs) have greater lateral elastic stiffness and lateral strength than other ones. Although the CBF has suitable lateral stiffness and strength, it suffers low dissipating energy capability in comparison with the other conventional systems. This weakness is because of the susceptibility of its compressive diagonal members to buckling. Buckling of the compressive diagonal members under cyclic loading degrades the system and reduces the capability of seismic energy dissipation [1,2]. To overcome this problem, ideas have been presented during the last decades, including reducing the slenderness ratio [3], utilizing the optimum configuration of the system [4,5], and strengthening the CBF using dissipating energy devices [6]. Additionally, researchers [7] confirmed that the slip between the slab and the beam interface contributed to the energy dissipated by the system, and the ductility demands decreased on other parts, such as the beam ends and the joints.

Using optimal configurations, as well as reducing the slender ratio of the diagonal member does not eliminate the degradation of the CBF system, although they improve the seismic performance of CBFs. Degradation of the stiffness and strength of the CBF system reduces dissipating energy. Among the presented ideas, utilizing dissipating energy devices is more suitable.

By using these devices in a structure, most of the input seismic energy dissipates in these devices. As a result, the damage is mainly limited to the devices [8]. Additionally, energy-absorbing devices reduce the seismic demand of structures [9]. Although these devices enjoy considerable advantages, especially in enhancing dissipation and the energy capability of the structures generally, they impose some additional costs on structural construction [10].

Comparing the energy dampers shows that passive dampers are cheaper and easier to fabricate than active and semi-active dampers. Among the available varieties of passive energy dampers, metallic dampers are one of the most effective and economical mechanisms for the dissipation of seismic energy input, which is achieved through the inelastic deformation of metallic material.

Numerous passive metallic dampers, as a device indirectly added to brace members, have been developed to improve the behavior of the CBF systems and are classified into two categories.

(a) The first category is dampers attached between the floor beam and CBF, including such steel plate-based dampers as added damping and stiffness (ADAS) [11,12], triangle ADAS (TADAS) [13,14], rhombic [15], X-shaped [16], slit [17,18], pre-bent strips [19] and curved steel dampers [20], and shear dampers [21,22]. These dampers have shown desirable seismic performance, but they need high-quality manufacturing. The main weakness of these dampers is the complexity of construction and fabrication. To overcome this shortcoming and employ their advantages, researchers have proposed shear links. Dampers made as shear links are easier to construct. The most famous links that act as dampers are EBF systems [23–25], vertical shear links [26,27], and shear dampers [28,29]. In addition to reducing the stiffness, the mentioned dampers have a suitable seismic performance and are easy to build.

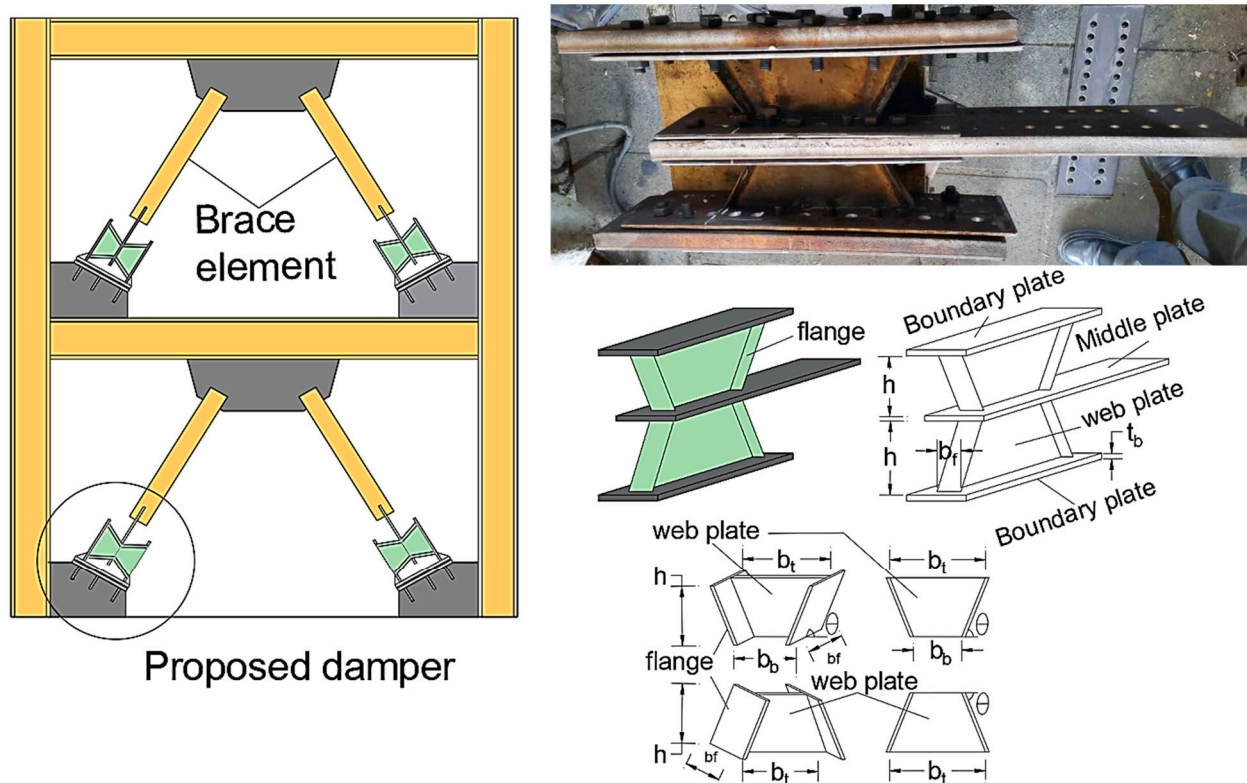
(b) The second category of dampers (directly added to the diagonal member) is more economical than indirectly attached dampers in CBF systems. Among the categorized dampers, the braced ductile shear panel [30], the U-Shaped steel [31], torsional beam [32], taper tube dampers [33], cushion damper [34], buckling-restrained brace (BRB) [35], semi-BRB [36], assembled bolt-connected buckling-restrained brace (AB-BRB) [37], novel external retrofitting sub-structure and self-centering precast bolt-connected steel-plate reinforced concrete buckling-restrained brace frame (SC-PBSPC BRBF) [38,39], U-shaped damper attached to diagonal brace member [40], X-shear damper [41], and box dampers [42] are generally known as the famous dampers.

Although the reviewed dampers enhance the seismic behavior of CBF systems, their manufacturing and implementation has some drawbacks. Generally, these problems cause constructors to use the mentioned dampers only in special buildings. Furthermore, they have no economic justification for conventional buildings, mainly short to moderate buildings [41,43]. In addition to the optimal seismic performance of structures, ease of implementation and economic considerations are other important issues that should be considered to ensure an efficient system. To overcome this problem, in this paper, an innovative passive metallic damper is introduced, which is easier to fabricate and install than the popular dampers such as ADAS, TADAS, viscous dampers, friction dampers, and BRB, which are presently being used worldwide. It is supposed that the damper prevents the nonlinear behavior of diagonal members. In other words, the damper limited the damage to predicted parts (elements of the damper), and other parts of the structures remained elastic. The behavior of the proposed damper is investigated numerically and parametrically. The main feature of this paper is improving the behavior of the I-shaped damper attached directly to the original CBF system using the same materials used for the dampers. The proposed damper is supposed to increase the ultimate strength and stiffness more than the I-shaped damper, which is investigated experimentally and numerically. Moreover, the required formula to design the proposed damper is presented.

## 2. The Trapezoidal Damper

### 2.1. Construction Detailing

Figure 1 illustrates the proposed damper. The damper is directly attached to the diagonal element member of the CBF system. As it is expected that the damper acts as a ductile fuse, it prevents buckling of the diagonal element member of the CBF system. Moreover, it can be easily replaced after a severe earthquake. The damper can be easily assembled and fabricated.

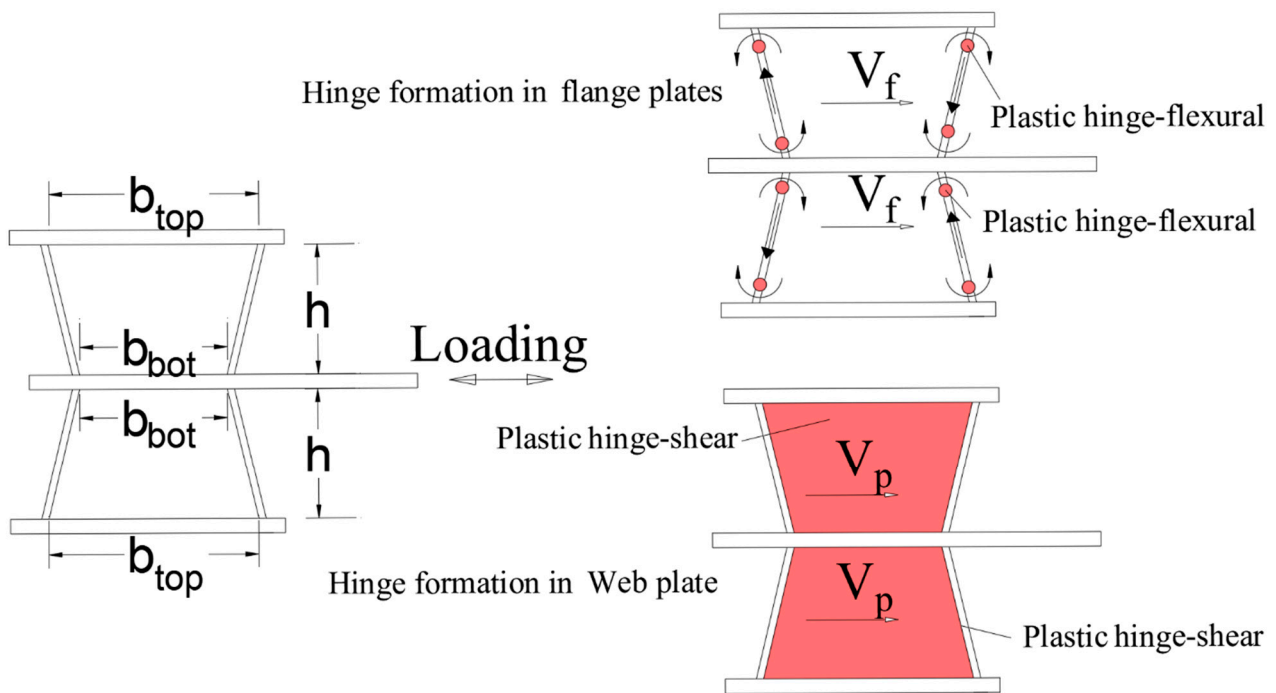


**Figure 1.** Construction details of the proposed damper.

The damper is made of a web of trapezoidal-shaped plates that are attached to two flange plates on two sides. Two of the created plates are connected to the middle plates; then, they are surrounded by two boundary plates. It is expected that the middle plates and boundary do not contribute to the load-bearing. Therefore, the imposed load on the damper is resisted by web plates and flanges plates. To create the trapezoidal web plates, plates with two ends angle of  $0 \leq \theta \leq 90$  can be used. As the angle of principal stress is  $45^\circ$ , therefore,  $\theta = 45^\circ$  is utilized for the proposed damper. It should be noted that using  $\theta = 90^\circ$ , an I-shaped link as a damper is shaped. The I-shaped damper has been investigated comprehensively in [43], which is used as a benchmark to compare with the trapezoidal damper.

### 2.2. The Behavior of the Proposed Damper

As pointed out in the previous section, the middle plate and boundary plates are expected to remain elastic. Accordingly, they do not contribute to resisting the applied loads. Additionally, imposed seismic energy is dissipated by the hinge formation in the web and flange plates. Accordingly, it is expected that the plastic hinges will be formed over the proposed damper, as illustrated in Figure 2. According to the figure, two flexural hinges are formed at the two ends of each flange plate, and a shear hinge is formed in each web plate.



**Figure 2.** Formed plastic hinges over the proposed damper.

According to the AISC 341-16 [44], for I-shaped shear links, when the  $\rho \leq 1.6$ , the shear capacity,  $V_n$ , of a shear link is calculated as  $V_n = V_p = 0.6F_{yw}bt_w$  where  $V_p$  is plastic shear capacity. In this equation, the capacity of the flange plates has been ignored. However, in this paper, the angled flange plates are accounted for in the shear capacity of the trapezoidal damper. Therefore, by accounting web plate and flange plate in determining the shear capacity of the damper, Equation (1) is proposed as:

$$V_n = 2(V_p + \epsilon V_f) \quad (1)$$

In this equation,  $V_f$  is the shear capacity of the flange plate. As the distributed load is applied to the flange using the web, the capacity of the flange decreases according to  $\epsilon = 0.75$ . So, for this damper, Equation (1) is simplified as  $V_n = 2V_p + 1.5V_f$ .

The behavior of the proposed damper consists of buckling and yielding under applied loading. In the fully shear yielding, the capacity of the proposed damper is reached  $V_n = n \frac{F_y}{\sqrt{3}} bt_p$ . The geometry of the proposed damper affected the damper the capacity. The strength of the proposed damper is measured by:

$$V_p = 2\sigma_{yx}b_{bot} \cdot t_p \quad (2)$$

where 2 is the number of main plates and the  $\sigma_{yx}$  is the stress on the main plate that is determined based on the buckling and yielding capacity of the main plates.

Based on Basler [45], elastic shear buckling stress,  $\tau_{cr}$ , is obtained from Equation (3). Since the basic materials are steel, by accounting, the Poisson ratio is  $\vartheta = 0.3$ .

$$\tau_{cr} = \frac{K \cdot \pi^2 \cdot E}{12(1 - \vartheta^2)} = 0.9KE \left( \frac{t_p}{b} \right)^2 \quad (3)$$

$$\begin{cases} K = 5.34 + 4(b/h)^2 & \frac{b}{h} \geq 1 \\ K = 4 + 5.34(b/h)^2 & \frac{b}{h} \leq 1 \end{cases} \quad (4)$$



Subsequently, the aspect ratio of the damper plate is low, and buckling that occurs in nonlinear zones also should be accounted for.

$$\tau_{cr} = 0.63 \frac{t_p}{b} \sqrt{KF_y E} \leq \frac{F_y}{\sqrt{3}} \quad (5)$$

The stresses in the main plate are established along the angle of  $\theta = 45^\circ$ . Accordingly, the stresses were as follows.

$$\sigma_{xx} = \sigma_{ty} \cdot \cos^2 \theta = 0.5 \sigma_{ty} \quad (6)$$

$$\sigma_{yy} = \sigma_{ty} \cdot \sin^2 \theta = 0.5 \sigma_{ty} \quad (7)$$

$$\sigma_{xy} = \sigma_{yx} = \tau_{cr} + 0.5 \sigma_{ty} \cdot \sin^2 \theta = \tau_{cr} + 0.5 \sigma_{ty} \quad (8)$$

where,  $\sigma_{ty}$  is equivalent yield stress. According to the von Mises yield criterion, the web plate's yielding occurs when  $(\sigma_{xx} - \sigma_{yy})^2 + (\sigma_{xx} - \sigma_{zz})^2 + (\sigma_{yy} - \sigma_{zz})^2 + 6(\sigma_{xy}^2 + \sigma_{yz}^2 + \sigma_{xz}^2) = 2F_y^2$ . Since Plan stress is formed in the web plate, the  $\sigma_z = \sigma_{yz} = \sigma_{xz}$  equal to zero. Considering  $\sigma_{xy} = \tau_{xy}$ , this equation is simplified as  $\sigma_{xx}^2 + \sigma_{yy}^2 + 6\sigma_{xy}^2 = 2F_y^2$ . Substituting Equations (6)–(8) into von Mises yield criterion equations give:

$$\sigma_{ty}^2 + 6(\tau_{cr} + 0.5\sigma_{ty})^2 = 2F_y^2 \quad (9)$$

Therefore, the value of  $\sigma_{ty}$  at which yielding of the plate occurs is calculated  $\sigma_{ty}$  is

$$\sigma_{ty} = \frac{-1.71\tau_{cr} \pm \sqrt{(1.71\tau_{cr})^2 - 0.9(3\tau_{cr}^2 - F_y^2)}}{2} \quad (10)$$

The shear capacity stress of the plate is given by Equation (11) for  $\tau_{cr} \leq \frac{F_y}{\sqrt{3}}$  and Equation (12) for  $> \frac{F_y}{\sqrt{3}}$ .

$$\sigma_{yx} = \left( \min \left[ 1.64h \frac{ht_p}{b} \sqrt{K \frac{F_y}{E}}, 2.34K \frac{h^2 t_p^2}{b^2} \right] + 0.5 \sigma_{ty} \right) \quad (11)$$

$$\sigma_{yx} = \tau_y = \frac{F_y}{\sqrt{3}} \tau_{cr} \quad (12)$$

Additionally,  $V_f$  is calculated from Equation (13). This equation has been derived based on the possibility of plastic hinge formation, as illustrated in Figure 2. This equation is simplified of  $\frac{4M_{pf}}{h} \sin \theta$  that corresponds to the formation of two flexural plastic hinges at the two ends of the flange plates where  $M_{pf}$  is the plastic moment of the flange plate.

$$V_f = \frac{\sqrt{2} b_f t_f^2}{2 h} F_{yf} \quad (13)$$

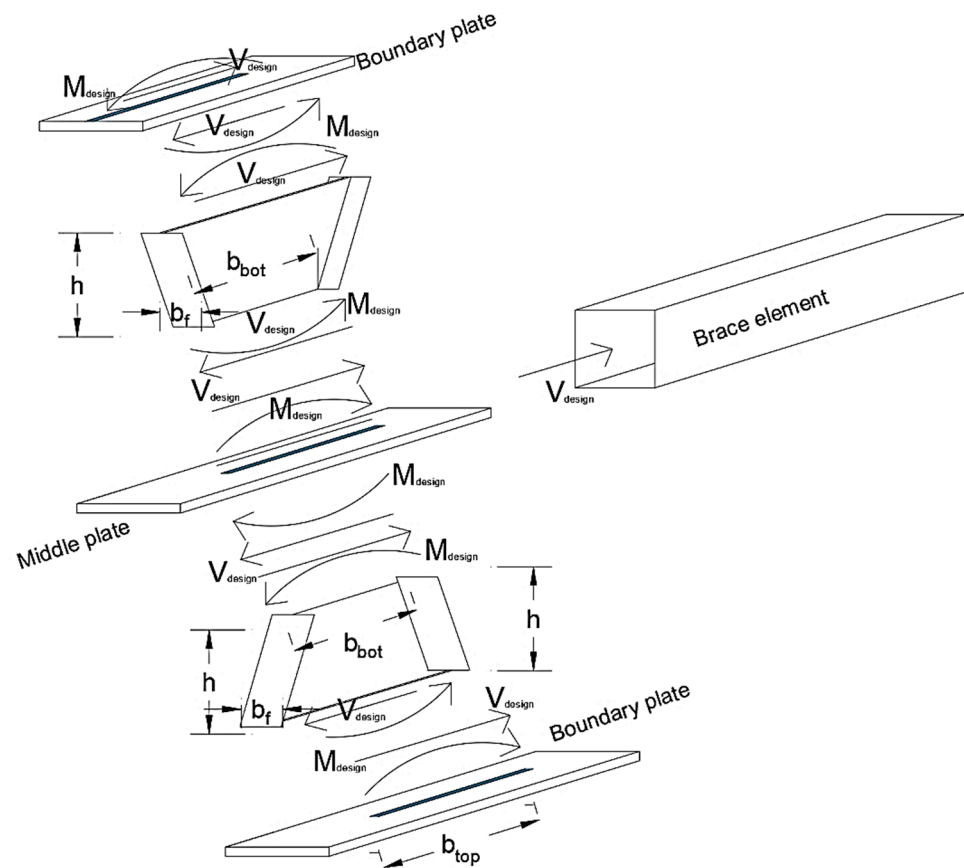
where  $F_{yf}$  is the yielding stress of the flange plate. Other parameters have been shown in Figure 1.

### 2.3. Design of the Damper and Elements Outside the Damper

To design the proposed damper, first, Equation (1) is used to design the elements (web and flange plates) of the damper against lateral loading. Substituting Equations (2) and (12) into Equation (1) and simplifying gives:

$$V \leq V_n = 2(0.6F_{yw}bt_w) + 1.5 \frac{\sqrt{2} b_f t_f^2}{2 h} F_{yf} \quad (14)$$

where  $V$  is the imposed axial forces to the diagonal brace element due to lateral loading. Therefore, according to Equation (14), the main and flange plates of the proposed damper are designed. Accordingly, the elements outside the damper are designed according to Equations (15) and (16), according to Figure 3. To assure the elements outside the proposed damper remain elastic, they should be designed for forces greater than the capacity of the damper. To do so, the elements are designed to resist against  $V_d$ , presented in Equation (15). Accordingly, first, the proposed damper is designed under forces presented in Equation (1), and the elements outside the damper are designed for the amplified capacity,  $V_d$ .



**Figure 3.** Free body diagram of the proposed damper.

The proposed approach to design the elements outside the damper follows AISC 341-16, nevertheless the overstrength,  $\Omega$  ( $\Omega$  is discussed in the next sections) and  $V_n$ , is proposed in this paper. In AISC341-16, a constant value of  $\Omega$  has been recommended to equal 1.5. Nevertheless, some studies [46,47] showed that  $\Omega$  is not constant for shear links.

$$V_d = \omega V_n \quad (15)$$

where  $\omega = \max(1.25R_y, \Omega)$ ,  $R_y$  is the ratio of expected to nominal yield stress that is measured according to the AISC 341-16. For this damper, it is suggested to  $R_y = 1.2$ . Subsequently, the boundary plates must, against compressive forces, equal  $V_d$  and moment of  $V_d h$ . Additionally, the diagonal element of braces are designed under the axial force of  $V_d$ .

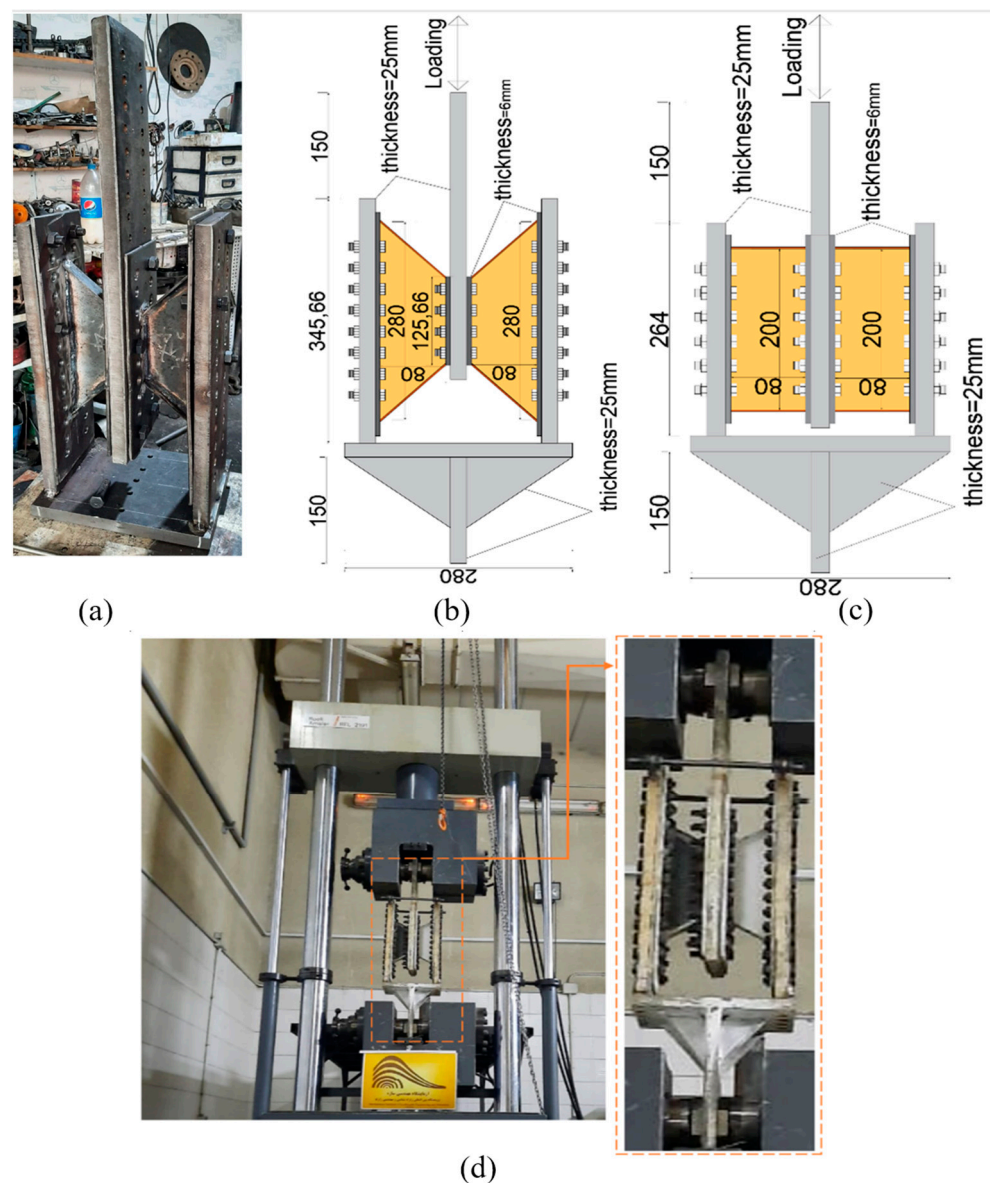
In so doing, the ultimate strength of the brace member,  $\phi P_n$ , where  $P_n$  is nominal resistance in compression should be satisfied, Equation (16). In this equation,  $\phi = 0.9$  and the  $P_r$  is the compression force to brace members due to the maxim strength of the damper.

$$\phi P_n \geq V_d \quad (16)$$

### 3. Experimental Study

#### 3.1. Preparing the Specimens

Two specimens, an I-shaped damper and a trapezoidal damper, illustrated in Figure 4, were prepared and tested under cyclic loading. To have a fair comparison, the same area section was designed for both dampers. Since the volume of materials used for both dampers is the same, by comparing their behavior, it is possible to evaluate which damper has better performance under completely identical conditions, which will indicate that it is more economical. The thickness of the web and flange plates was selected as 2 mm. Accordingly, the boundary plates and middle plates were designed according to Equation (14), which are plates with a thickness of 25 mm. To examine the capability of the damper's easy replacement, 6-mm plates were connected to the top and bottom of the main web and flange plates then they were bolted to the middle and boundary plates. The distance between the bolts was designed according to the AISC341-16.



**Figure 4.** (a) Prepared Trapezoidal specimen before testing, (b) Trapezoidal specimen detail) I-shaped damper detail, (c,d) Test preparing trapezoidal damper.

For all elements, ST37 steel was used with yielding stress = 235, ultimate strength = 370 MPa, Poisson's ratio = 0.3, and Young's modulus = 200 GPa. For bolts, M12 (diameter of 12 mm) and materials of ST59 steel with yielding stress of 590 MPa were used.

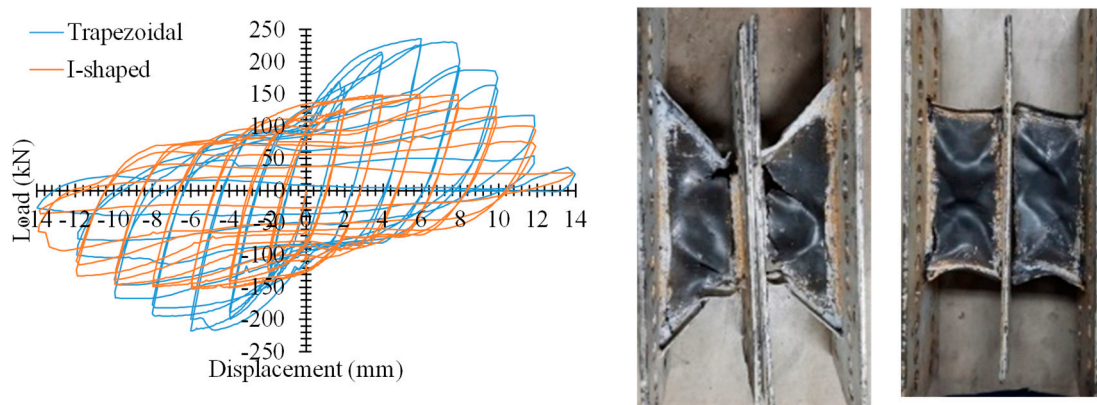
### 3.2. Boundary Condition

The test setup is shown in Figure 4d. The specimens were tested at the International Institute of Earthquake Engineering and Seismology, and the hysteresis curve was automatically captured using the lab equipment. The gusset plate of the damper was fixed at the bottom of the damper, and the actuator applied the defined load cyclically at the top. The loading was measured as displacement control and applied based on AISC-16. Before applying the cyclic loading, a monotonic analysis was performed to determine the yield point displacement ( $\Delta y$ ), then it was repeated (three times for each cycle) as  $\mp\Delta y$ ,  $\mp 2\Delta y$ , etc.

### 3.3. Experimental Results

#### 3.3.1. Specimen Performance

In Figure 5, the hysteresis curves of the tested specimens are compared. The results of this figure indicate that both dampers have good hysteresis curves with stable performance. As shown in this figure, the trapezoidal damper has better performance in ultimate strength, stiffness, and greater dissipating energy. A small pinching is observed in the hysteresis curves. This pinching is due to hinge formation in the flange and its buckling during the test. These parameters are compared in the next section.



**Figure 5.** Experimental test results.

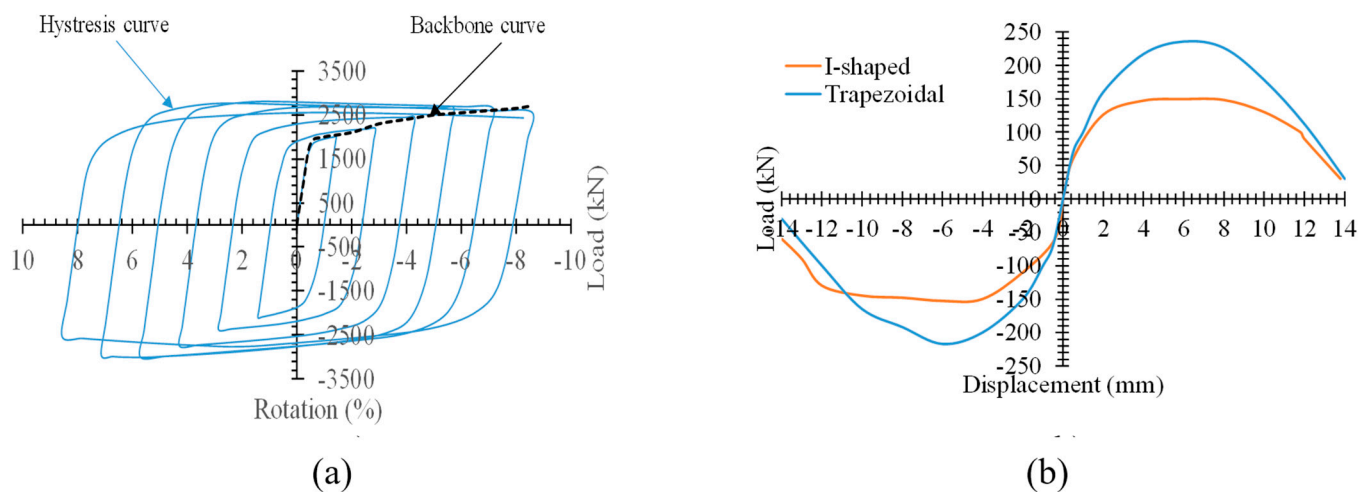
In both the dampers, web and flange plates experience nonlinear behavior, whereas middle plates and boundary plates remain elastic. This behavior confirms the accuracy of Equation (14) in the design of elements outside the damper.

In the trapezoidal specimen, the flange plate buckling and web plate cracking along the tension field occurred at a displacement of 8 mm (rotation = 10% or 0.1 rad). At a displacement of 10 mm (rotation = 12.5% or 0.125 rad), considerable cracking of the web plate appeared, followed by complete cracking at 12 mm (rotation = 15% or 0.15 rad).

For the I-shaped specimen, the crack propagation was initiated at the location where the web plate was connected to the flange plate. At the displacement of 10 mm (rotation = 12.5% or 0.125 rad), the flange was torn. The damper was resistant up to the displacement of 14 mm (rotation = 17.5% or 0.175 rad); as a result, the web plate tore at the vicinity of the connection to the middle plate. Although in AISC, the ultimate rotation of the shear link is limited to 8% or 0.8 rad, for the proposed damper, no rupturing/cracking occurred for any specimen, even up to a rotation of 10% or 0.1 rad (8 mm). This confirms the ductile behavior of the damper. As none of the bolts was fractured even up to the cracking of the damper, it can be confirmed that the damper can be easily replaced after a severe earthquake

### 3.3.2. Structural Parameters

The structural parameters of the models are obtained from the ideal bilinear curve, which is obtained from the envelope curves (backbone curve), as shown in Figure 6; the envelope curves are (backbone curve) derived according to FEMA-356. The parameters include elastic stiffness ( $K$ ), displacement corresponding to yielding ( $\Delta_y$ ), ductility ( $\mu = \Delta_{\max}/\Delta_y$ ), force corresponding to the first hinge formation ( $V_s$ ), and  $V_n$  that corresponds to the maximum/nominal shear strength of the damper; the structural parameters are listed in Table 1. Additionally, the backbone curve helps to find a better comparison of the results. As the hysteresis curve of the dampers is symmetrical, by extracting the backbone curve, the results of the hysteresis curve can be achieved at high accuracy. The schematic view of the backbone curve of the dampers is plotted in Figure 6.



**Figure 6.** Skeleton curve of the dampers (a) schematic view (b) extracted for dampers.

**Table 1.** Structural parameters of the damper.

Specimen	$V_n$ (kN)		$K$ (kN/mm)	$\Delta_y$ (mm)		$\mu$		$\Omega$
	+	−		+	−	+	−	
Trapezoidal	235.76	217.3	141.32	1.67	1.54	7.19	4.68	2.05
I-shaped	149.22	152.8	130.58	1.14	1.17	8.75	7.48	1.31
T/I	1.58	1.42	1.08	1.46	1.32	0.82	0.63	1.56

$V_n$ : strength,  $K$ : Stiffness,  $\Delta_y$ : displacement corresponding to the yielding,  $\mu$ : ductility, and  $\Omega$ : overstrength.

Referring to the results, the parameters improved as ultimate strength up to 58%, stiffness around 8%, overstrength 56%, and displacement corresponding to yielding up to 46%, but the ductility is reduced by around 37%. The parametric study is carried out in the next sections to investigate other factors on the structural parameters of the dampers. Additionally, the ultimate strength was calculated by Equation (1) which gave 240.22 kN. It predicted the ultimate strength with +2% and −10% errors.

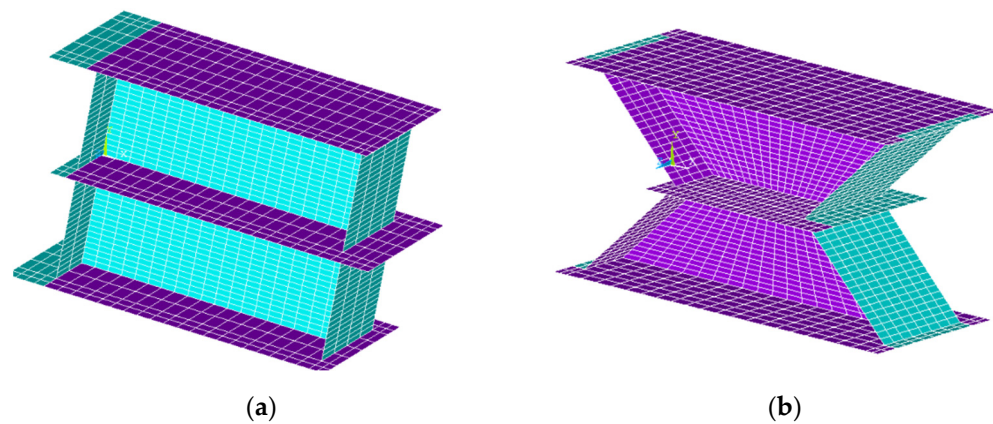
## 4. Numerical Study

### 4.1. Finite Element (FE) Modeling

In this study, the finite element (FE) simulation was carried out using the ANSYS program. For modeling the damper, all elements were simulated utilizing SHELL 181 with six degrees of freedom in each node. This element has the capability to measure large displacement, buckling, and material nonlinearity. Although solid elements (brick elements) could also be used, since the analysis of the models using shell elements takes less time and leads to the same results as solid elements, SHELL 181 was used.

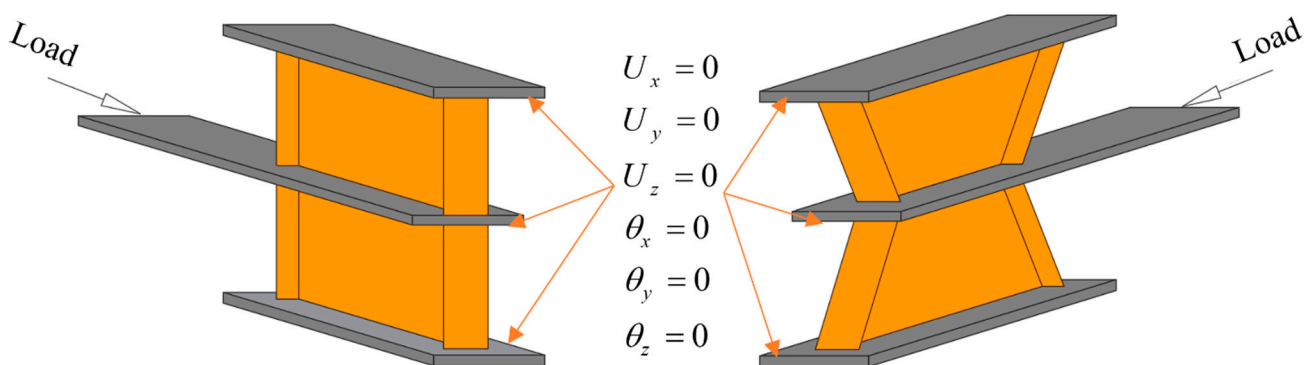


The FE models, as illustrated in Figure 7, were meshed so that each damper is composed of 2328 elements.



**Figure 7.** The view of FE modeling of the dampers to show the mesh sizing (a) I-shaped (b) Trapezoidal.

The boundary condition of the FE models was applied, as shown in Figure 8. Since the end of the damper attached to the gusset plate acts as a rigid connection, its end models as fixed supported. The other end of the damper, where it is connected to the brace member, is simulated as a simple connection, and loading is applied. This determined boundary condition for FE modes is similar to the situation of a real damper without modeling the diagonal element member. The loading was displacement control; it was increased until the amount of drift (rotation) of web plates reached 8% (0.08 radians). Although experimental results have proven that the proposed damper can withstand a rotation of more than 8%, the results of numerical studies suggest that the maximum rotation based on AISC 341-16 is 8%. As indicated in article F3.4a of AISC 341-16, short/shear links attain a maximum rotation of 0.08 rad under seismic loading. Therefore, the maximum displacement of 11.2 mm was applied to the damper to reach the rotation of 0.08 rad.



**Figure 8.** Boundary conditions for FE simulation.

#### 4.2. Verification of FE Results

In Figure 9, the FE results are compared with experimental test results. As shown in this figure, the FE results are in good agreement with the test results. The FE modeling captures the stiffness of the dampers' math with the test results in elastic and inelastic zones. However, lower strength using FE is obtained in comparison with the test results.

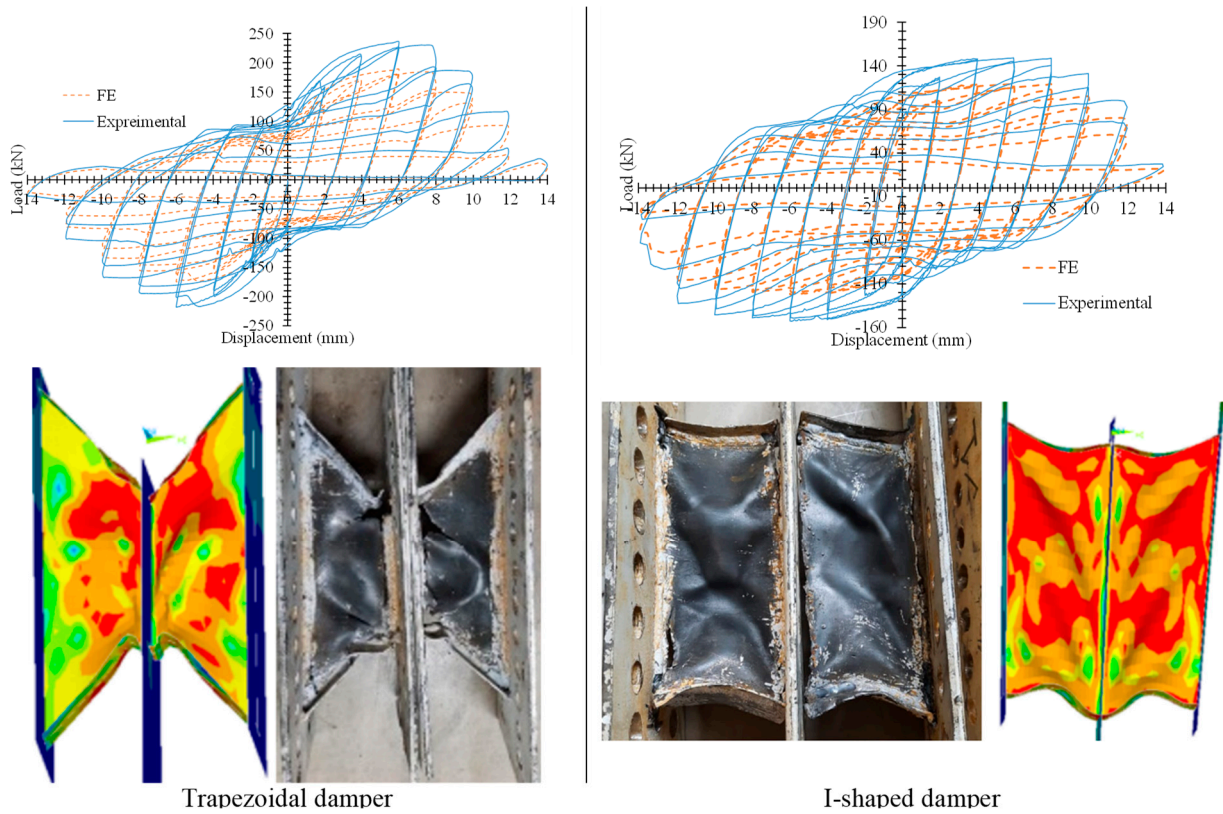


Figure 9. Tested specimens.

5. Numerical Results

5.1. Hysteresis Curves

Figure 10 illustrates the hysteresis curves of the I-shaped damper and trapezoidal damper. To show the overall cyclic behavior of these dampers, dampers with minimum and maximum  $t_f$  have been compared. As shown in this figure, either damper pertains to stable hysteresis curves without degradation. Moreover, increasing the  $t_f$  cause improves the hysteresis curves of the dampers.

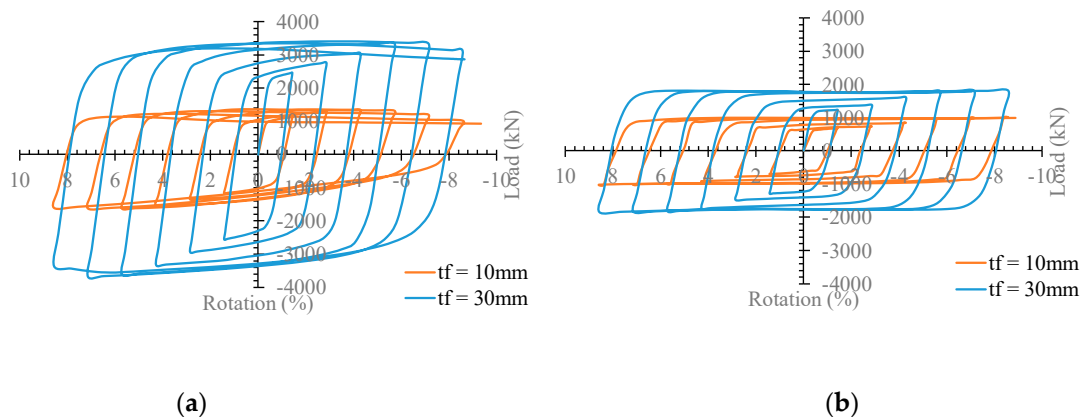
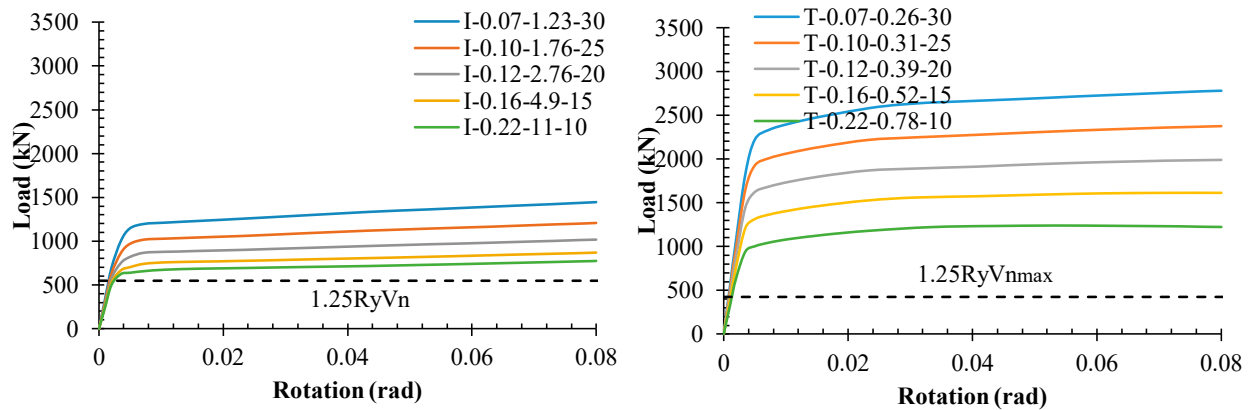


Figure 10. Comparing the hysteresis curves of the dampers (a) I-shaped (b) Trapezoidal.

To show better performance and since the dampers have a symmetrical hysteresis curve, their skeleton curves (backbone curves) are discussed in the next section. To do so, in Figure 11, the load–rotation curves of the damper have been drawn. Referring to the figure provides that by increasing the  $t_f$ , the load–rotation curves of either damper tend to be enhanced, causing the curves to increase the structural parameters such as elastic stiffness,

ultimate strength, and dissipating energy, which is investigated in the next sections. The noticeable problem is that the amplified factor,  $1.25 R_y$ , in the design of the elements outside the damper is less than the ultimate strength. For trapezoidal damper, the  $V_{n,max}$  has been calculated according to the  $b_{top}$  that is the maxim  $b$  for the damper. Therefore, estimating the maxim load to the design of elements outside the damper according to the AISC-360-16 is unconservative. To do so, the overstrength related to the dampers should be used.



**Figure 11.** The load-rotation curves of the dampers.

### 5.2. Comparing the I-Shaped Damper with the Trapezoidal Damper

In Figure 12, the load rotation of the I-shaped damper with the trapezoidal damper is compared. As shown in this figure, the trapezoidal damper has a better performance than the I-shaped damper in the case of ultimate strength, elastic stiffness, and dissipating energy. According to the comparison in this figure, the load–rotation curve of the trapezoidal damper with  $t_f = 10$  mm is close to the I-shaped damper with  $t_f = 25$  mm. It is shown that trapezoidal dampers with less material can achieve a closer behavior than the I-shaped damper. The structural parameters are listed in Table 2.

**Table 2.** Comparing the structural parameters of the dampers.

Model	$V_n$ (kN)	K (kN/mm)	$V_s$ (kN)	$\Omega$	E (kN.mm)	$\mu$	Trapezoidal I-shaped						
							$V_n$	K	$V_s$	$\Omega$	E	$\mu$	
I-0.22-11-10	770.02	1779.74	498.26	1.55	7813.62	25.89							
I-0.16-4.9-15	869.93	1889.04	528.86	1.64	8811.98	24.32							
I-0.12-2.76-20	1015.11	2041.52	571.56	1.78	10,238.44	22.52							
I-0.10-1.76-25	1206.48	2242.91	627.94	1.92	12,100.72	20.82							
I-0.07-1.23-30	1447.04	2486.40	696.11	2.08	14,407.72	19.24							
T-0.22-0.78-10	1237.57	2289.51	639.04	2.09	13,029.37	20.72	1.61	1.29	1.28	1.35	1.67	0.80	
T-0.16-0.52-15	1615.99	2756.09	771.25	2.10	16,906.77	19.10	1.86	1.46	1.46	1.27	1.92	0.79	
T-0.12-0.39-20	1991.61	3170.87	887.70	2.24	20,648.02	17.83	1.96	1.55	1.55	1.26	2.02	0.79	
T-0.10-0.31-25	2377.29	3544.10	992.23	2.40	24,511.24	16.70	1.97	1.58	1.58	1.25	2.03	0.80	
T-0.07-0.26-30	2781.74	3885.19	1087.75	2.56	28,572.31	15.64	1.92	1.56	1.56	1.23	1.98	0.81	

$V_n$ : Ultimate strength, K: Stiffness,  $V_s$ : forces corresponding to the first yielding to the yielding, E: Energy dissipation,  $\Omega$ : overstrength.  $\mu$ : ductility.

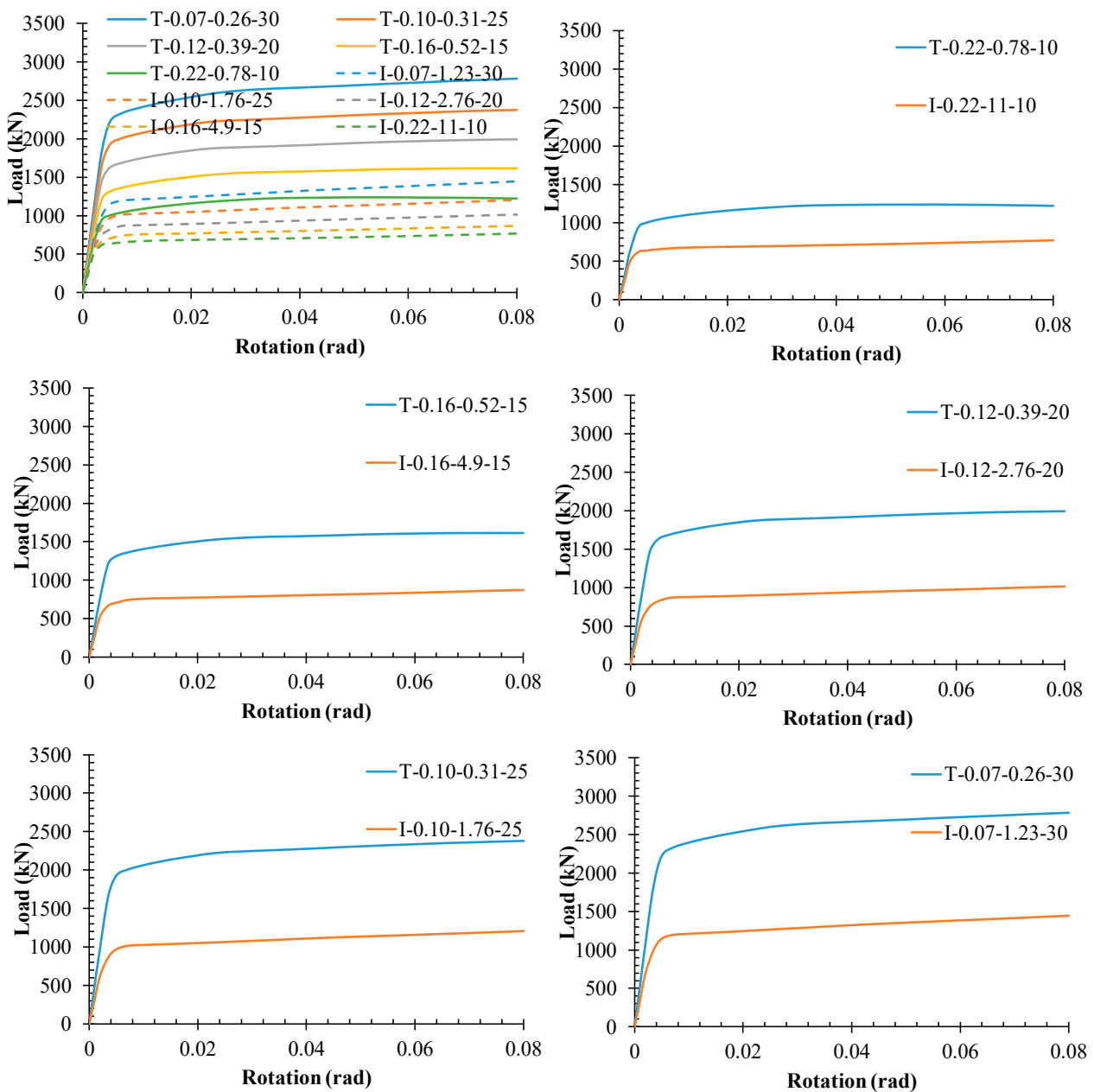


Figure 12. Comparing the load–rotation curves of the dampers.

Results indicated that using a trapezoidal damper instead of an I-shaped damper increased the ultimate strength ( $V_n$ ) between 61% and 92%, elastic stiffness ( $K$ ) between 29% and 56%, and the energy dissipating ( $E$ ) between 1.67 and 2.03 times, forces corresponding to the first yielding ( $V_s$ ) between 28% and 56% that is similar to the  $K$ . Additionally, overstrength ( $\Omega$ ) was enhanced between 23% and 35%, but by increasing the  $t_f$ , its rate was reduced. Moreover, the ductility ( $\mu$ ) was reduced by around 20%.

In Figure 13, the ratio of the parameters for the trapezoidal damper to the I-shaped damper versus  $t_f$  is plotted. This figure indicates that the most increasing due to trapezoidal damper divide by I-shaped damper is related to, respectively,  $E$ ,  $V_n$ ,  $K$ , and  $\Omega$ . This figure also shows the rate of increasing the parameters versus  $t_f$ .

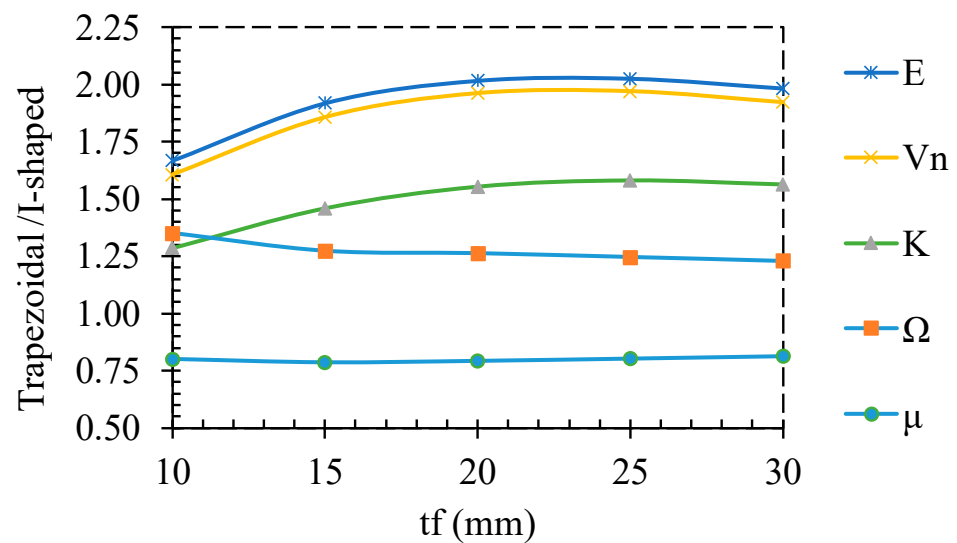


Figure 13. Ratio of parameters of the trapezoidal damper to I-shaped damper versus  $t_f$ .

### 5.3. Stiffness

Comparing the trapezoidal damper with the I-shaped damper indicates that the trapezoidal has greater elastic stiffness than the I-shaped. Additionally, the stiffness in the nonlinear zone is plotted versus rotation in Figure 14. This figure shows that the stiffness of the damper coincides together around a rotation of 0.006 Rad. In the other words, although the types of damper (I-shaped or trapezoidal) and thickness of the flange plate are effective on the stiffness, they do not have a considerable effect on the stiffness after rotation more than 0.006 Rad.

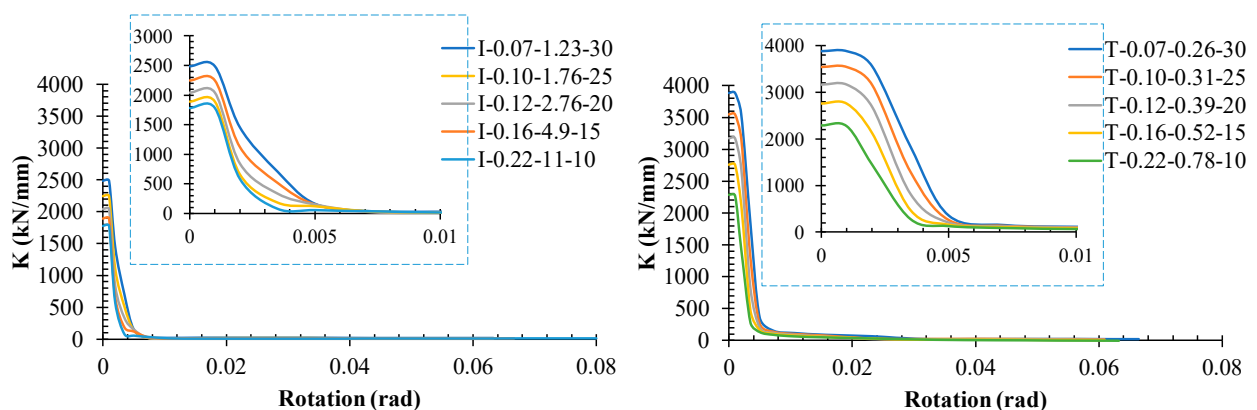


Figure 14. Stiffness of the dampers versus rotation.

As is expected, the stiffness contains the stiffness of the flange plate and web plate (main plate), the stiffness of the elements is plotted in Figure 15. As shown in this figure, for both dampers, more stiffness is provided by web plates than flange plates, but, by increasing the  $t_f$ , the effect of the flange plate is enhanced. Additionally, the effect of the flange plate in the trapezoidal damper is greater than the I-shaped damper. For the trapezoidal damper, although the stiffness of the web plate is greater than the flange plate, they coincide together at rotation around 0.002 Rad. It is concluded that after emerging yielding in the shear damper, the stiffness of the web plate is reduced, but the flange plate prevents more reduction.



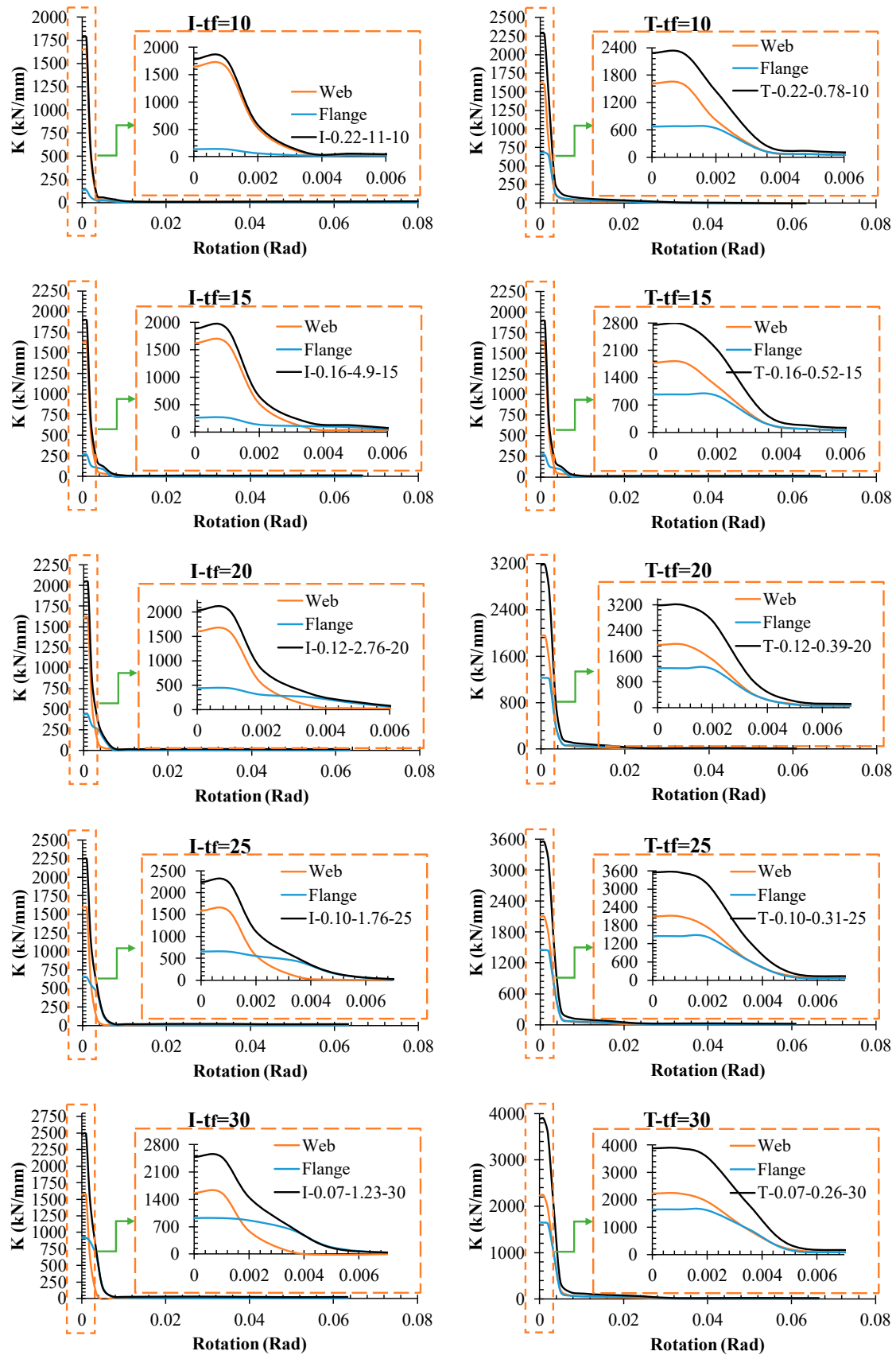
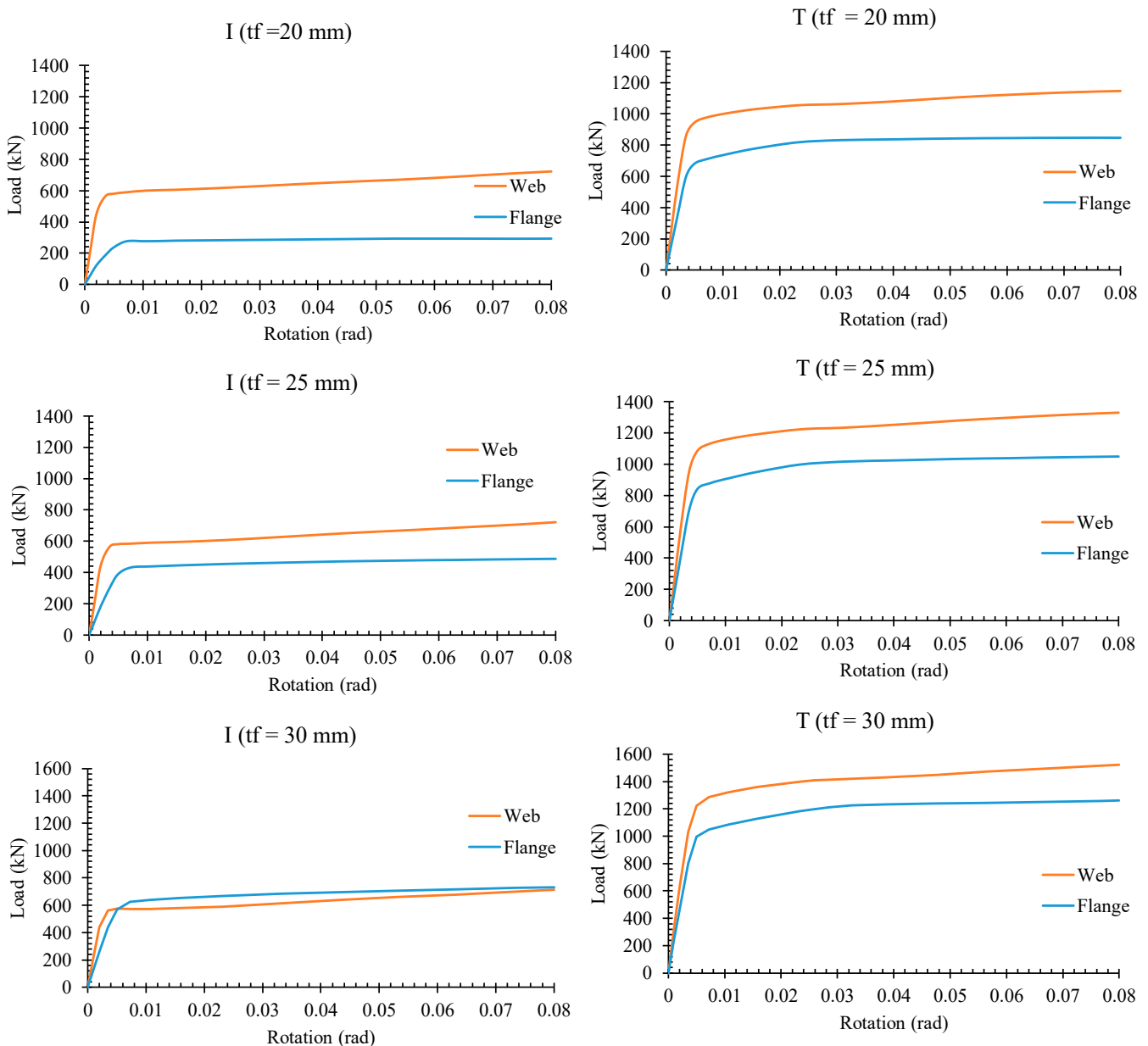


Figure 15. Stiffness versus rotation in case of web and flange effect.

#### 5.4. Share of Flanges on the Applied Load

According to the AISC-341-16, the flange plate in calculating the shear capacity of shear links is ignored. To consider the effect of the flange plate on the shear capacity of the dampers, the load–rotation for the web and flange plate for the damper is plotted in Figure 16. For a summarized and better showing, the I and T in these figures represent the trapezoidal damper and I-shaped damper, respectively. The  $t_f$  is given in the Perrantes. According to the figure, the flange plate contributes to load bearing, which confirms the assumption used in this paper to derive the equations govern to the damper’s behavior.



**Figure 16.** Load versus rotation for web plate and flange plate.

The I-shaped, thin flange plate has a lower effect in load bearing, and its effect is increased by increasing the  $t_f$ , whereas in the I-shaped damper with  $t_f = 30$  mm, the contribution of the flange plate and web plate is approximately the same. Nevertheless, flange plates in the trapezoidal damper have a greater effect on the contribution of load bearing than I-shaped damper. It is due to the angle of the flange plates. This is the same

as the I-shaped damper; the  $t_f$  causes an increase in the contribution of the flange plate in load bearing. However, even for the high, thick flange plate for the trapezoidal damper, the contribution of the flange plate is lower than the web plate.

To compare the percentage of the share of flange plate and web plate in load bearing, the share of the elements in the capacity of the damper is shown in Figure 17. Referring to the figure, at the beginning of applied loading, most of the loads are resisted by the web plate. Due to hinge formation in the web plate around the rotation of 0.006 Rad, its share in the capacity of the dampers is reduced. Consequently, the share of web plates tended to reduce, and the share of flange plates tended to rise.

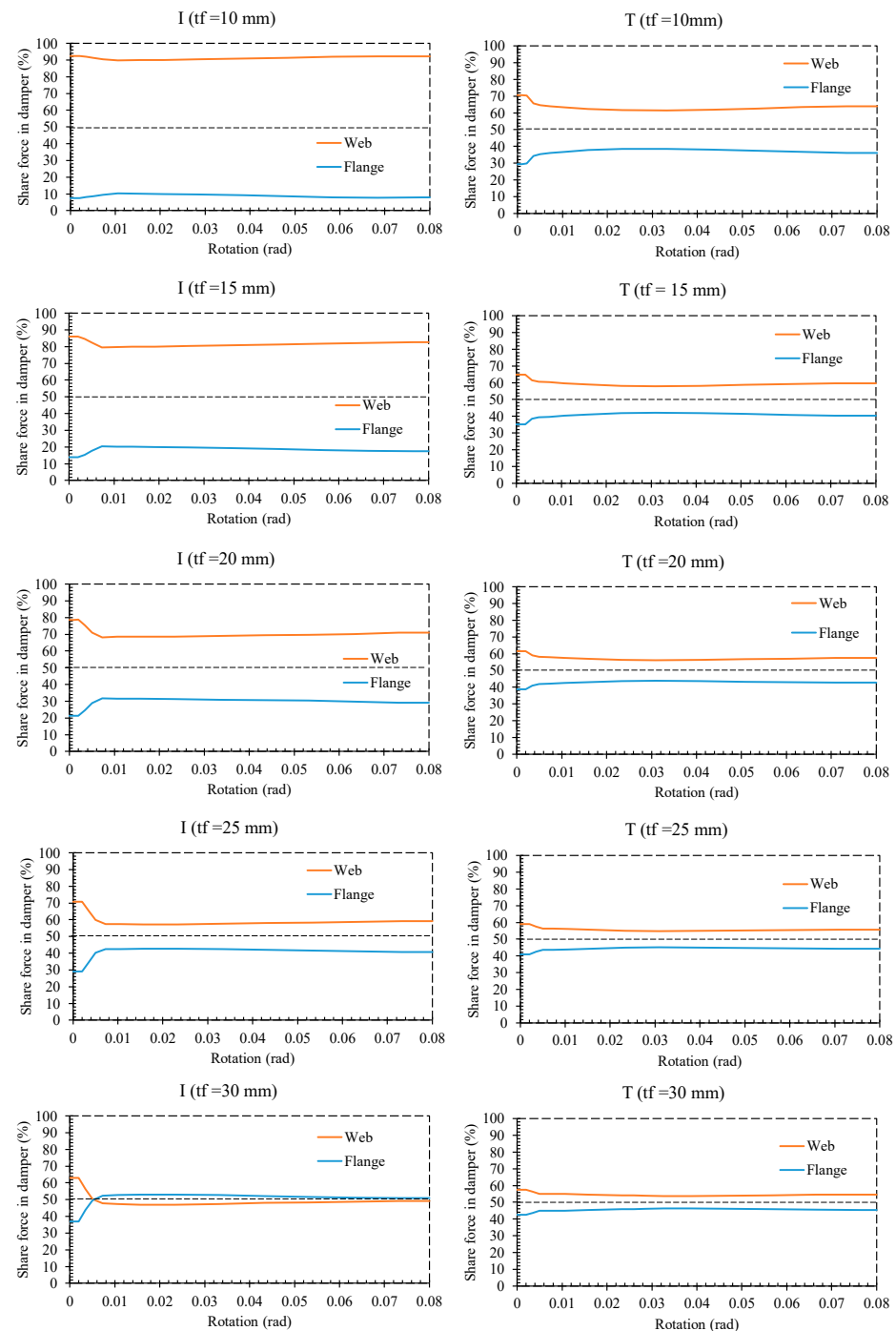
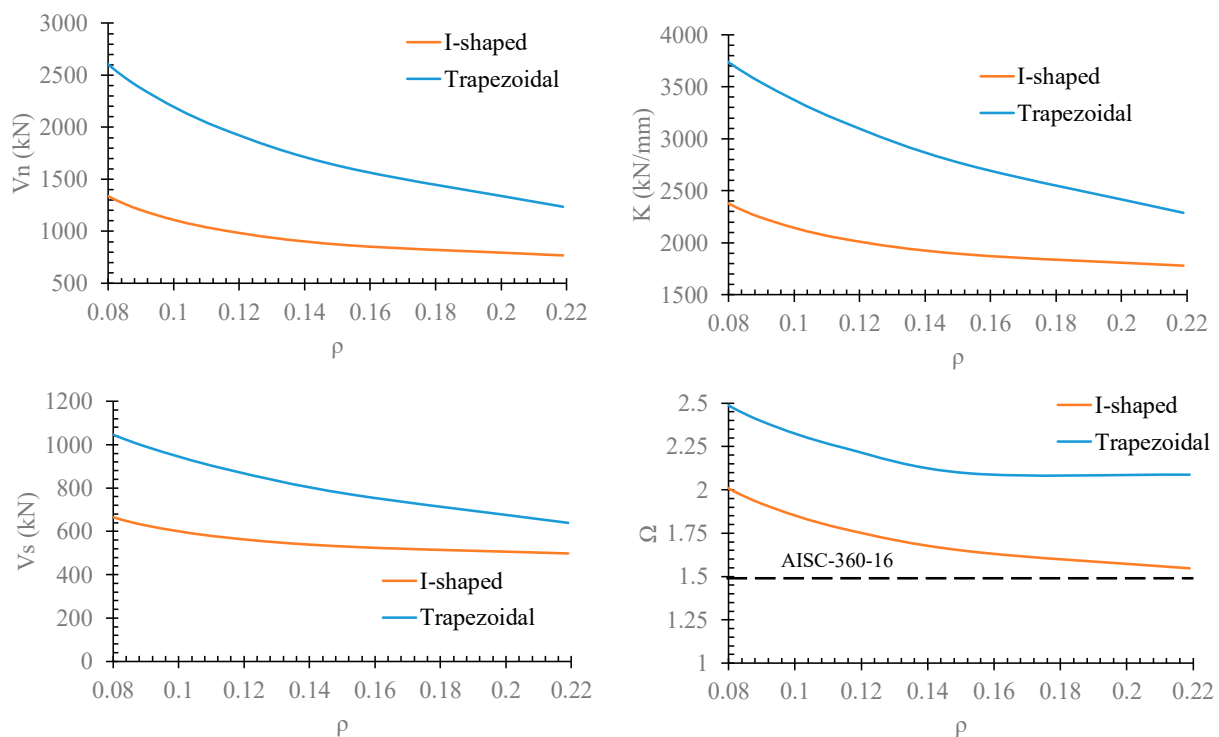


Figure 17. Share of flange plate and web plate for dampers.

For the I-shaped damper, 92.5% of the capacity of the damper is provided by the web plate at the beginning of the loading when a thin flange is used (7.5% by flange plates). By increasing the rotation, it is reduced to 89% (11% by flange plates). However, this is 70% at the beginning of the applied load and 62% after hinge formation in the web plate for the trapezoidal damper. By increasing the  $t_f$  from 10 mm to 30 mm, the share of web plate in load bearing at the beginning and after hinge formation in the damper, respectively, changed from 70% and 62% (30% and 38% for flange plates) to 57% and 55% (43% and 45% for flange plates).

### 5.5. Effect of $\rho$

According to the AISC-314-16, shear links with  $\rho < 1.6$  gives the shear capacity of  $V_p$ . In so doing, the structural parameters versus  $\rho$  are plotted in Figure 18 to consider the effect of  $\rho$  on the behavior of dampers. Referring to the figure, even though the dampers are designed to be very short links,  $\rho$  affects the results even if smaller than 1.6. By increasing the  $\rho$ , the structural parameters are reduced. However, the rate of reduction for the trapezoidal damper is greater than for the I-shaped damper. In the other words, the trapezoidal damper is more sensitive than the I-shaped damper. For  $0.12 < \rho > 0.25$ , the effect of  $\rho$  on the I-shaped damper dampers is ignorable.



**Figure 18.** The effect of  $\rho$  on the behavior of dampers.

### 5.6. Effect of $t_f$

To consider the effect of  $t_f$  on the behavior of dampers, the structural parameters of dampers with different  $t_f$  divided by damper with  $t_f = 10$  mm versus  $t_f$  are plotted in Figure 19. As shown in this figure, the  $t_f$  increases (from 10 mm to 30 mm; three times) the  $V_n$ ,  $V_s$ ,  $K$ , and  $\Omega$  up to 1.88, 1.4, 1.4, and 1.35 times for the I-shaped damper, respectively. Nevertheless, for the trapezoidal damper, the  $t_f$  increases the mentioned parameters, respectively, up to 2.25, 1.7, 1.7, and 1.23. Except for the  $\Omega$ , the effect of  $t_f$  on the structural parameters of the trapezoidal is greater than the I-shaped damper. It is concluded that the effect of flange plate thickness is not ignorable on the performance of the shear dampers, especially shear dampers with angled flange plates.

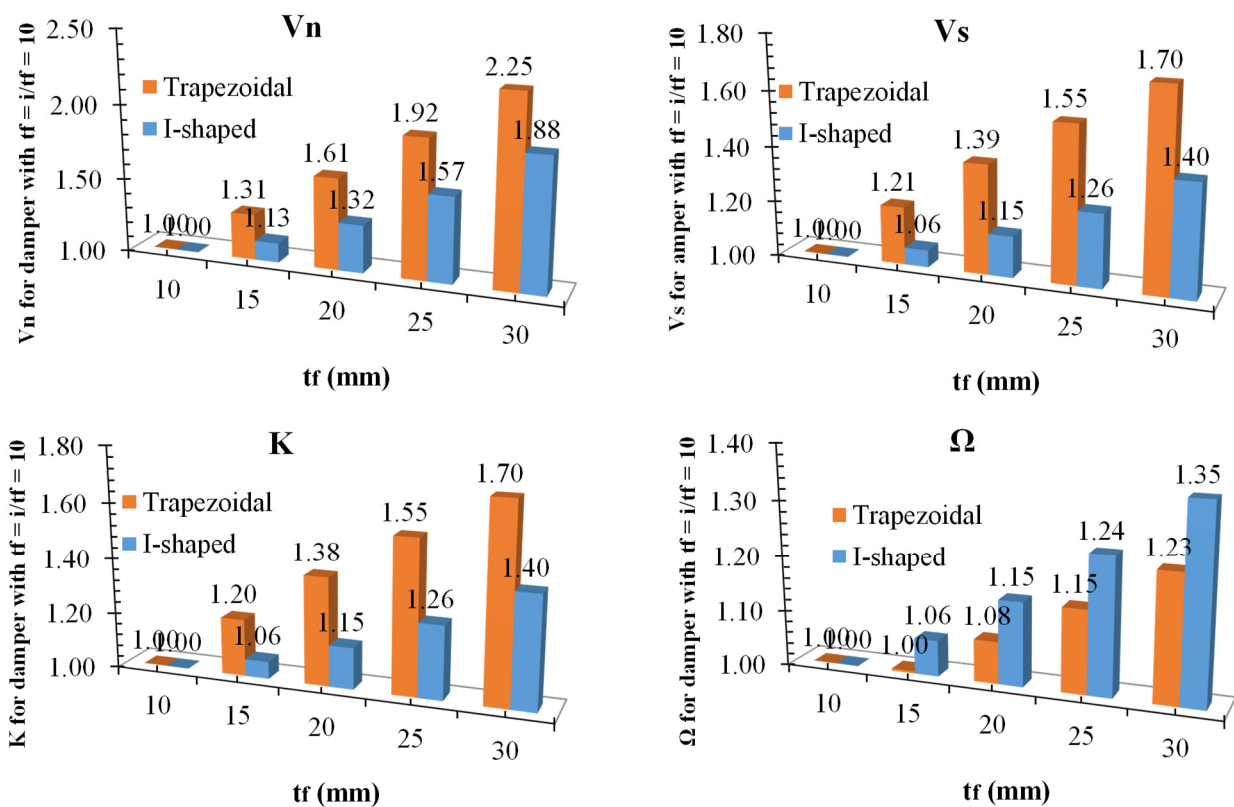


Figure 19. Structural parameters of dampers versus  $t_f$ .

Comparing the effect of  $t_f$  on the parameters  $V_n$ ,  $K$ , and  $\Omega$  reveals that the  $t_f$  has the most effect on the  $V_n$  for both dampers, Figure 20. However, it has the same effect on  $K$  and  $\Omega$  for the trapezoidal damper. In an I-shaped damper, the effect of  $t_f$  on the  $\Omega$  is more than  $K$ .

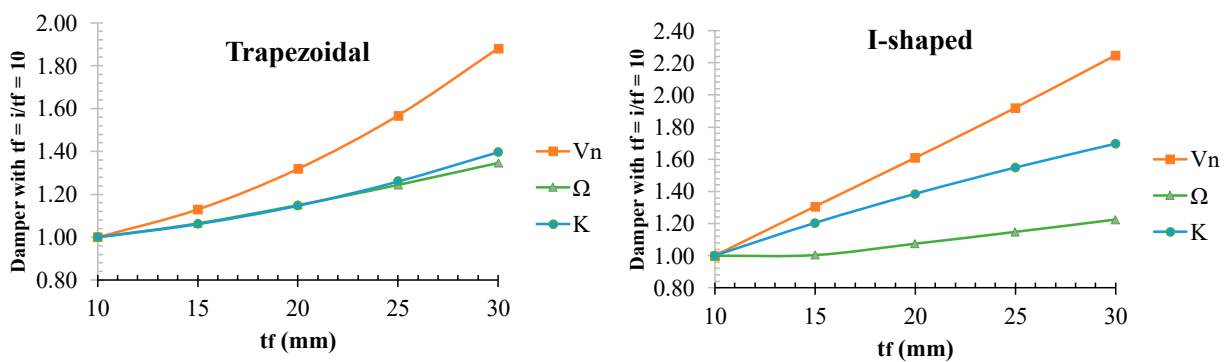


Figure 20. Comparing the structural parameters of dampers versus  $t_f$ .

### 5.7. Effect of $\psi$

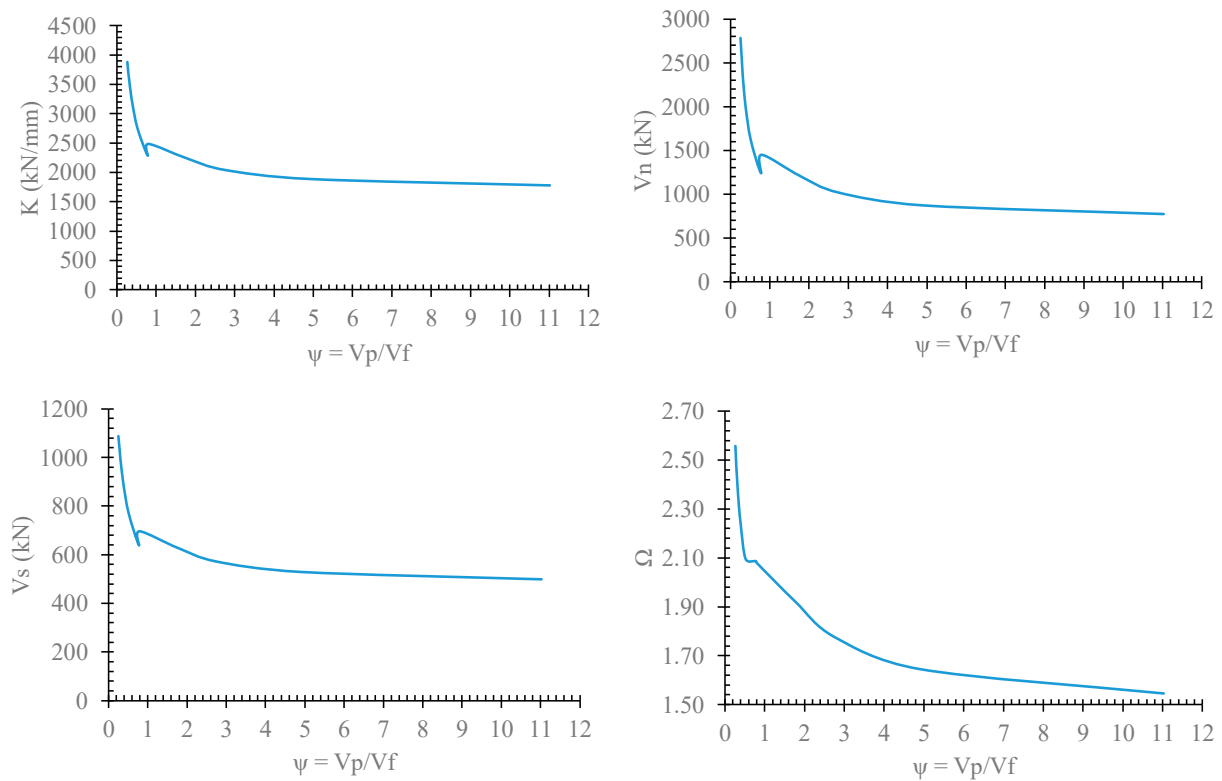
In addition to  $t_f$ , the non-dimension parameter,  $\psi$ , affects the behavior of the damper with different values of  $\theta$ . As shown in Figure 21, if  $\psi$  increases, the strength, stiffness, overstrength, and load corresponding to the first hinge formation decrease. The rate of reduction can be divided into regions with  $\psi < 1$  and  $\psi > 1$ . The rate of reduction of the mentioned parameters with  $\psi < 1$  is higher than that in  $\psi > 1$ . Therefore, it is suggested to



use dampers with  $\psi < 1$ . Using  $\psi < 1$ , we obtain  $V_p < V_f$ . Equation (12) should be satisfied based on the flange thickness.

$$t_f \geq 0.3 \frac{F_{yw}}{F_{yf}} \frac{b_{bot}}{b_f} \frac{t_w}{\cos \theta} \quad (17)$$

$$t_f \geq \sqrt{0.6 \frac{h}{b_f} \frac{F_{yw}}{F_{yf}} \frac{t_w b_{bot}}{\sin \theta}} \quad (18)$$



**Figure 21.** Nominal strength versus  $\psi$ .

### 5.8. Overstrength

Since the elements outside the dampers are related to the  $\Omega$  factor, calculating the  $\Omega$  is important. In so doing, the  $\Omega$  of the FE models is listed in Table 3. Additionally, the  $\Omega$  versus  $t_f$ ,  $\rho$ , and  $\psi$  are plotted in Figure 22 to measure the effect on the parameters on the  $\Omega$ .

**Table 3.** The  $\Omega$  of the FE models.

Model	$t_f$ (mm)	$\rho$	$\psi$	$\Omega$
I-0.22-11-10	10	0.22	11.03	1.55
I-0.16-4.9-15	15	0.17	4.90	1.64
I-0.12-2.76-20	20	0.13	2.76	1.78
I-0.10-1.76-25	25	0.1	1.76	1.92
I-0.07-1.23-30	30	0.07	1.23	2.08
T-0.22-0.78-10	10	0.22	0.78	2.09
T-0.16-0.52-15	15	0.16	0.52	2.10
T-0.12-0.39-20	20	0.12	0.39	2.24
T-0.10-0.31-25	25	0.10	0.31	2.40
T-0.07-0.26-30	30	0.07	0.26	2.56

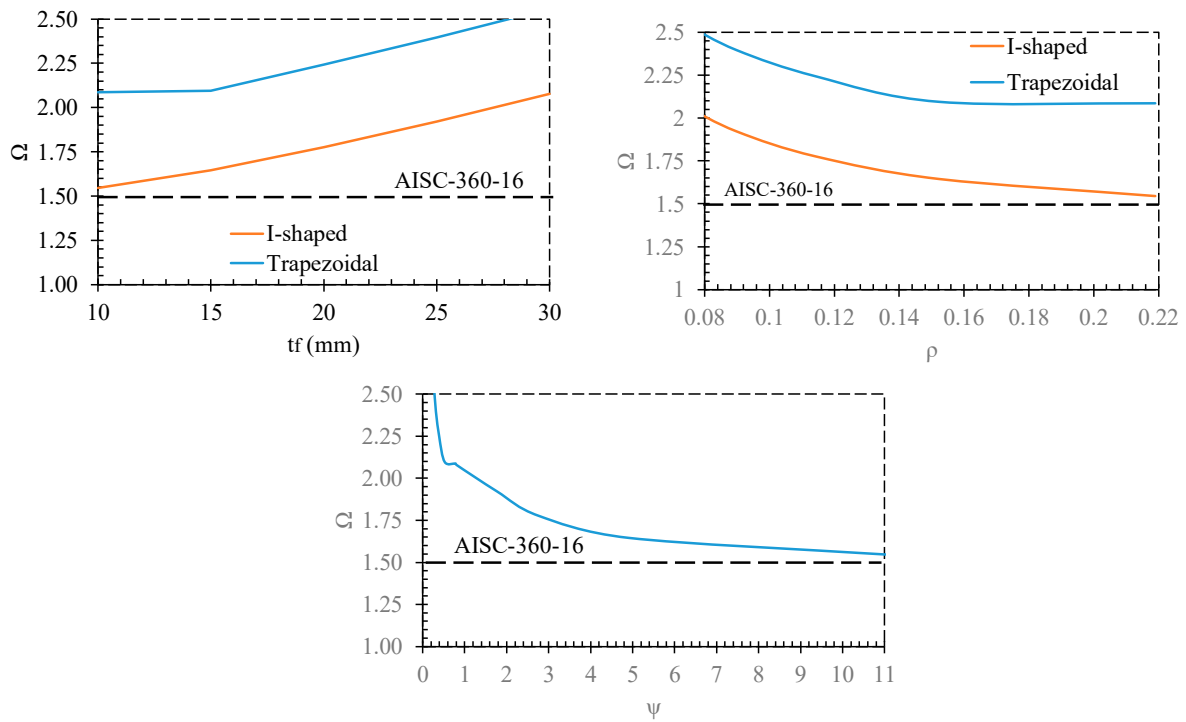


Figure 22. Considering the  $\Omega$ .

According to Figure 22, the proposed dampers that were accounted as shear links reveal a  $\Omega$  more than AISC. In addition to the problem, the  $\Omega$  is increased by the increase in  $t_f$ , a reduction in  $\rho$ , and a reduction in  $\psi$ . Therefore, giving a constant confession for  $\Omega$  is not logical.

As confirmed, the  $\Omega$  is affected by all the  $t_f$ ,  $\rho$ , and  $\psi$ . Since the  $\Omega$  implicitly includes the conditions of the  $t_f$ ,  $\rho$ , therefore, Equation (19), which is derived based on  $\psi$ , is suggested to determine the value of  $\Omega$ . This equation is derived based on the fitting of finite element results. The proposed equation and FE results are compared in Figure 23, which shows a good agreement.

$$\Omega = 2 - 0.25 \ln \psi \tag{19}$$

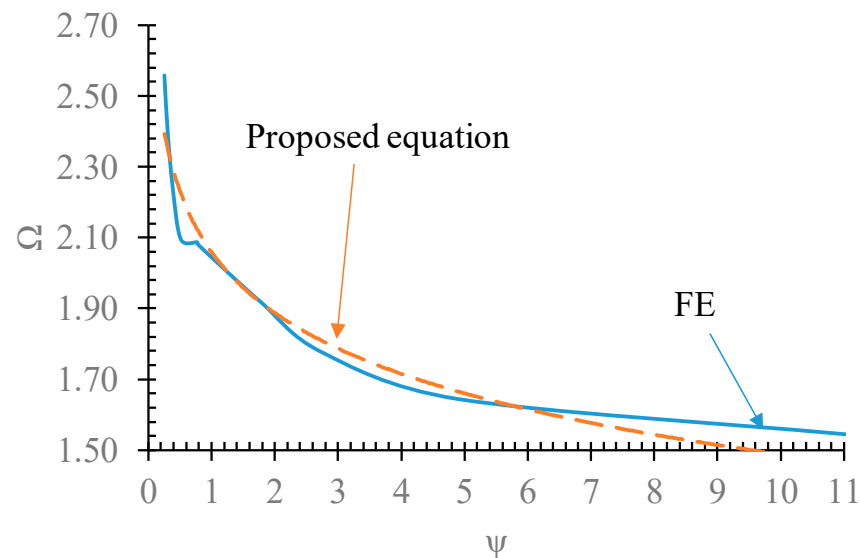


Figure 23. Comparing the  $\Omega$  determined by the proposed equation and FE results.

### 5.9. State of Yielding

Figure 24 illustrates the yielding state of the damper at the rotation of 0.08 Rad. As shown in this figure, the hinge formations in these dampers are in good agreement with the assumption presented in Figure 2. For both dampers, by increasing the flange thickness, the yielding had better distribution over the web plate. Additionally, the middle plate and boundary plates remained elastic. Therefore, it is concluded that the proposed equation presented in the previous section to design elements outside the main elements of the damper is capable. It is expected to remain elastic in the elements outside the web plate and flange plate.

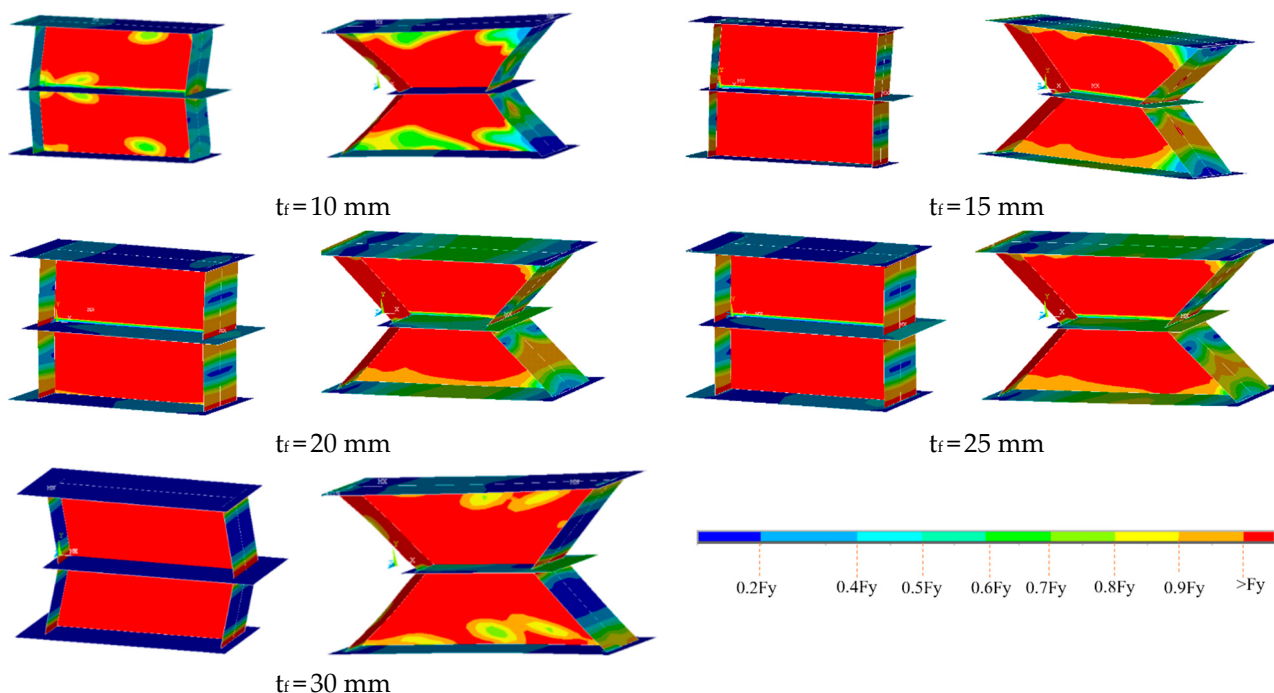


Figure 24. Yielding state over the dampers.

### 5.10. Accuracy of the Proposed Damper Equations

In Table 4, the results of proposed relations with FE results are compared. As presented in this table, the proposed equations are in good agreement with FE results.

Table 4. Comparing the proposed equations with FE results.

	$V_{pmin}$ (kN)	$V_f$ (kN)	$V_{min}$ (kN)	FE (kN)	Error
T-0.22-0.78-10	1260	12.93	1915.86	1237.57	+55
T-0.16-0.52-15	1260	29.09	1948.18	1615.99	+21
T-0.12-0.39-20	1260	51.71	1993.42	1991.61	0.00
T-0.10-0.31-25	1260	80.80	2051.60	2377.29	−14
T-0.07-0.26-30	1260	116.35	2122.70	2781.74	−24

## 6. Conclusions

In this paper, an innovative shear damper with a trapezoidal shape was proposed to improve the behavior of I-shaped dampers and was investigated experimentally and numerically. The proposed damper is easy to fabricate and easy to replace after a severe earthquake. Also, the required equations for the design of the paper were presented. The results are summarized as follows.

- The experimental and numerical study indicated that the trapezoidal dampers have greater ultimate strength, elastic stiffness, dissipating energy, and overstrength than I-shaped.
- Although the flange plate in both dampers affected their behavior, the thicknesses of the flange plate in the trapezoidal damper have more effect on the behavior of the damper than the I-shaped damper.
- The amplified factor,  $1.25 R_y$ , to the design of the elements outside the damper is less than the ultimate strength. Therefore, to design elements outside the damper, using  $\omega = \max(1.25 R_y, \Omega) V_n$  is suggested. In this relation, the overstrength is proposed as  $\Omega = 2 - 0.25 \ln \psi$ .
- Numerical results indicated that when using a trapezoidal damper instead of an I-shaped damper, the ultimate strength is between 61% and 92%, elastic stiffness (K) between 29% and 56%, and the energy dissipating (E) between 1.67 and 2.03 times are improved.
- The stiffness of the damper coincides together around the rotation of 0.006 Rad. In the other words, although the types of damper (I-shaped or trapezoidal) and thickness of the flange plate are effective on the stiffness, they do not have a considerable effect on the stiffness after rotation more than 0.006 Rad.
- For an I-shaped damper, 92.5% of the capacity of the damper is provided by a web plate at the beginning of the loading when a thin flange is used (7.5% by flange plates). By increasing the rotation, it is reduced to 89% (11% by flange plates). However, this is 70% at the beginning of the applied load and 62% after hinge formation in the web plate for the trapezoidal damper.
- By increasing the  $t_f$  from 10 mm to 30 mm, the share of web plate in load bearing at the beginning and after hinge formation in the damper, respectively, change from 70% and 62% (30% and 38% for flange plates) to 57% and 55% (43% and 45% for flange plates).
- Comparing the effect of  $t_f$  on the parameters  $V_n$ , K, and  $\Omega$  reveals that the  $t_f$  has the most effect on the  $V_n$  for both dampers. However, it has the same effect on K and  $\Omega$  for the trapezoidal damper. In an I-shaped damper, the effect of  $t_f$  on the  $\Omega$  is more than K.
- Recommendations for future work: To complete the research in this field, it is recommended to investigate the effect of the proposed damper to improve the behavior of reinforced concrete (RC) systems, especially for existing structures. Since the damper improves the behavior of the CBF system, can be built easily, and does not impose much cost on a structure, a comprehensive study is needed to achieve an optimum configuration between the damper and RC frame.

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