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# Mixed Convection Boundary Layer Flow over a Horizontal Circular Cylinder in $Al_2O_3$ –Ag/Water Hybrid Nanofluid with Viscous Dissipation

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#### **ARTICLE INFO**

#### ABSTRACT

#### Article history:

Received 13 July 2023 Received in revised 15 August 2023 Accepted 17 September 2023 Available online 1 January 2024 This paper investigated the mathematical modelling for mixed convection boundary layer flow over a horizontal circular cylinder in  $Al_2O_3$ –Ag/water hybrid nanofluid with viscous dissipation. The transformed partial differential equations (PDEs) are numerically solved using an implicit finite-difference approach known as the Keller-box method. The numerical solutions for the reduced Nusselt number,  $Nu_xRe_x^{-1/2}$ , local skin friction coefficient,  $C_fRe_x^{1/2}$ , temperature profile,  $\theta(y)$  and velocity profiles f'(y) are found and graphically presented in detail. Effects of the Eckert number, Richardson number and nanoparticle volume fraction are all examined and explained. It is found that the increase of volume fraction of nano material in nanofluid has increased the value of skin friction coefficient. The low density of nano oxides such as alumina in hybrid nanofluids also contribute to reduce friction between fluid and body surface. Based on numerical analysis, the combination of nanoparticles in the form of  $Al_2O_3$ –Ag/water hybrid nanofluid may reduce skin friction phenomena while sustaining heat transfer characteristics comparable to Ag/water nanofluid. The results in this paper are original and will assist researchers working in the field of boundary layer flow. It can also be utilised as a reference in experimental studies to reduce operating costs.

## Keywords:

Mixed convection; hybrid nanofluid; circular cylinder; viscous dissipation

## 1. Introduction

Nanotechnology is used in many engineering applications to dissipate heat. The primary objective is to improve heat transfer in engineering devices such as heat exchangers and engine cooling systems. The thermal conductivity of fluids commonly used in heat transfer, such as water and ethylene glycol, is considered inadequate at both the fluid and air surfaces. As a result, the heat transfer rate in a car radiator must be increased, requiring the development of innovative heat transfer fluids [1]. However, Nano-coolants are a new class of coolants developed as a result of the most recent developments in science [2]. In 1995, Choi from Argonne National Laboratory introduced

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the concept of "nanofluids", which indicate the stable suspension produced by suspending metal, metallic oxide, or non-metallic nanoparticles with average diameters of less than 100 nm in a base fluid [3]. Nanofluid has potential in thermal management systems such as liquid-submerged cooling for transformers and electronic circuit boards, due to its successful engine downsizing implementation in the automotive industry [4].

Metal nanoparticles have been recognized for their superior heat transfer capabilities compared to oxide nanoparticles, due to their thermal conductivity, density, and specific heat characteristics. The use of nanofluids has been experimentally proven to enhance many thermal properties, including thermal conductivity, thermal diffusivity, viscosity, and convective heat transfer, as compared to conventional base fluids like water and oil [5]. As a lubricant additional, nanoparticle additives like CuO (copper oxide),  $MoS_2$  (molybdenum disulfide), and  $TiO_2$  (titanium dioxide) can be used to minimize friction and wear [6]. Using a water-based fluid, three distinct types of nanoparticles are considered: Cu,  $Al_2O_3$ , and  $TiO_2$ . It is shown that an increase in the volume fraction of each distinct nanoparticle results in a corresponding rise in both the skin friction coefficient and heat transfer rate at the surface [7].

Numerous investigations have been done by researchers to examine the movement of fluids within a circular cylinder, Tham *et al.*, [8] and Mohamed *et al.*, [9] who have investigated mixed convection over a horizontal circular cylinder have seen an increase in the skin friction coefficient at the middle of the cylinder (1.1-1.5 radians) before falling towards the cylinder's end surface. Based on a review of the literatures and the opinions of certain researchers, focus on Newtonian heating by Salleh *et al.*, [10], the flow of Jeffrey fluid by Zokri *et al.*, [11], flow past a symmetric cylinder by Mahat *et al.*, [12], nanofluid by Tham *et al.*, [13], flow of viscoelastic fluid by Widodo *et al.*, [14]. Following that, the flow and heat transfer are immersed in a viscoelastic nanofluid by Mahat *et al.*, [15]. The results indicate that an increase in the mixed convection parameter results to an increase in skin friction values, while Nusselt number values exhibit the opposite trend.

Recent research has been conducted on the phenomenon of fluid flow on a horizontal circular cylinder in a hybrid nanofluid. Notably, the works of Roy and Akter have contributed to this field of research [16] who reported that  $Al_2O_3$ —Cu/water in the form of hybrid nanofluid increases heat transfer by approximately 28.28% when compared to  $Al_2O_3$ —water in the form of nanofluid and by approximately 51.15% when compared to pure fluid. The properties of a hybrid nanofluid including two nanoparticles CuO and either Au or Al differ significantly from those of a nanofluid containing only one component CuO by Alwawi et al., [17].

Based on the articles mentioned above that have been reviewed, It is noteworthy to mention that convective heat transfer on a circular cylinder is still an interesting topic to explore. Therefore, the purpose of the present study is to investigate the mixed convection boundary layer flow over a horizontal circular cylinder in  $Al_2O_3$ –Ag/water hybrid nanofluid with viscous dissipation. The governing Partial Differential Equations (PDEs) are solved using numerical methods, and the investigation of the fluctuation of relevant physical parameters has not been previously conducted, hence, the findings presented in this study are original.

## 2. Mathematical Formulations

The object under consideration is a horizontal circular cylinder (HCC) with a given radius a, which is heated to a constant temperature  $T_w$  embedded in a hybrid nanofluid with ambient temperature  $T_\infty$ . Figure 1 show the physical representation of the case. The orthogonal coordinates of  $\bar{x}$  is measured along the cylinder surface. Starting from the lower stagnation point  $\bar{x}=0$ , and  $\bar{y}$  calculates the distance normal to the surface. The boundary layer approximation is valid, the dimensional

governing equations of steady mixed convection boundary layer flow are taken from the previous study [18, 19]:

$$\frac{\partial \overline{u}}{\partial \overline{x}} + \frac{\partial \overline{v}}{\partial \overline{y}} = 0, \tag{1}$$

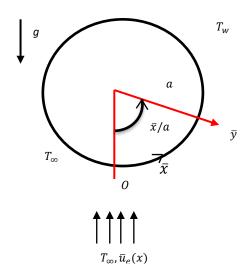
$$\bar{u} \frac{\partial \bar{u}}{\partial \bar{x}} + \bar{v} \frac{\partial \bar{u}}{\partial \bar{y}} = \bar{u}_e \frac{d\bar{u}_e}{d\bar{x}} + \frac{\mu_{hnf}}{\rho_{hnf}} \frac{\partial^2 \bar{u}}{\partial \bar{y}^2} + \frac{(\rho \beta)_{hnf}}{\rho_{hnf}} g(T - T_{\infty}) \sin \frac{\bar{x}}{a}, \tag{2}$$

$$\bar{u} \frac{\partial T}{\partial \bar{x}} + \bar{v} \frac{\partial T}{\partial \bar{y}} = \frac{k_{hnf}}{(\rho c_p)_{hnf}} \frac{\partial^2 T}{\partial \bar{y}^2} + \frac{\mu_{hnf}}{(\rho c_p)_{hnf}} \left(\frac{\partial \bar{u}}{\partial \bar{y}}\right)^2, \tag{3}$$

subjected to the boundary condition:

$$\bar{u}(\bar{x},0) = \bar{v}(\bar{x},0) = 0, \quad T((\bar{x},0) = T_w,$$
 (4)

$$\bar{u}(\bar{x},\infty) \rightarrow \bar{u}_e, \quad T(\bar{x},\infty) \rightarrow T_\infty$$



**Fig. 1.** Physical model of the coordinate system for mixed convection [20]

Where  $\bar{u}$  and  $\bar{v}$  represent the velocity components along the  $\bar{x}$  and  $\bar{y}$  axes, respectively.  $\bar{u}_e$  is external velocity. The symbol  $\mu_{hnf}$  represents the dynamic viscosity of a hybrid nanofluid, while  $\rho_{hnf}$  denotes the density of the hybrid nanofluid. The variable g represents the acceleration due to gravity,  $\beta_{hnf}$  represents the thermal expansion coefficient of the hybrid nanofluid, T represents the local temperature,  $\left(\rho C_p\right)_{hnf}$  is the heat capacity of hybrid nanofluid,  $v_{hnf}$  is the kinematic viscosity of hybrid nanofluid and  $k_{hnf}$  is the thermal conductivity of hybrid nanofluid which can be expressed from the previous study [18, 21, 22]:

$$v_{hnf} = \frac{\mu_{hnf}}{\rho_{hnf}}, \qquad \mu_{hnf} = \frac{\mu_f}{(1 - \phi_1)^{2.5} (1 - \phi_2)^{2.5}},$$

$$\rho_{hnf} = (1 - \phi_2) [(1 - \phi_1)\rho_f + \phi_1\rho_{s1}] + \phi_2\rho_{s2}$$

$$(\rho\beta)_{hnf} = (1 - \phi_2)[(1 - \phi_1)(\rho\beta)_f + \phi_1(\rho\beta)_{s1}] + \phi_2(\rho\beta)_{s2}$$

$$\left(\rho C_p\right)_{hnf} = (1 - \phi_2) \left[ (1 - \phi_1) \left(\rho C_p\right)_f + \phi_1 \left(\rho C_p\right)_{s1} \right] + \phi_2 \left(\rho C_p\right)_{s2},$$

$$\frac{k_{hnf}}{k_{bf}} = \frac{k_{s2} + 2k_{bf} - 2\phi_2(k_{bf} - k_{s2})}{k_{s2} + 2k_{bf} + \phi_2(k_{bf} - k_{s2})}, \qquad \frac{k_{bf}}{k_f} = \frac{k_{s1} + 2k_f - 2\phi_1(k_f - k_{s1})}{k_{s1} + 2k_f + \phi_1(k_f - k_{s1})}$$

The subscript  $_{hnf,~f,~s1}$  and  $_{s2}$  denote the physical characteristics of a hybrid nanofluid, base fluid, alumina  $Al_2O_3$  nanoparticle, and silver Ag nanoparticle, respectively. In this study, initially 0.06 vol. solid nanoparticle of Ag ( $\phi_2$ = 0.06) is added into water-based fluid to form Ag/water nanofluid. In the meantime, 0.1 vol. solid nanoparticle of  $Al_2O_3$  ( $\phi_1$ = 0.1) is added to the Ag/water nanofluid to create the  $Al_2O_3$ -Ag/water hybrid nanofluid.

The governing dimensionless variables are presented:

$$x = \frac{\overline{x}}{a}, y = Re^{\frac{1}{2}} \frac{\overline{y}}{a},$$

$$u = \frac{\overline{u}}{u_{\infty}}, v = Re^{\frac{1}{2}} \frac{\overline{v}}{u_{\infty}},$$
(5)

$$\theta(\eta) = \frac{T - T_{\infty}}{T_{w} - T_{\infty}}, \qquad \bar{u}_{e}(x) = u_{\infty} sin\left(\frac{\bar{x}}{a}\right), \qquad u_{e}(x) = \frac{\bar{u}_{e}(x)}{u_{\infty}} = \sin x.$$

Using Eq. (5), Eq. (1) - Eq. (3) becomes

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0,\tag{6}$$

$$u\frac{\partial u}{\partial x} + v\frac{\partial u}{\partial y} = u_e \frac{du_e}{dx} + \frac{v_{hnf}}{v_f} \frac{\partial^2 u}{\partial y^2} + \frac{(\rho\beta)_{hnf}}{\rho_{hnf}\beta_f} \lambda\theta \sin x,\tag{7}$$

$$u\frac{\partial\theta}{\partial x} + v\frac{\partial\theta}{\partial y} = \frac{k_{hnf}}{v_f(\rho c_p)_{hnf}} \frac{\partial^2\theta}{\partial y^2} + \frac{v_{hnf}}{v_f} \frac{\rho_{hnf}(c_p)_f}{(\rho c_p)_{hnf}} Ec \left[\frac{\partial u}{\partial y}\right]^2, \tag{8}$$

Subject to boundary conditions:

$$u(x,0) = 0, \quad v(x,0) = 0, \quad \theta(x,0) = 1,$$
 (9)

$$u(x,\infty) \to u_e, \ \theta(x,\infty) \to 0$$

Where  $\theta$  is the rescale dimensionless temperature of the hybrid nanofluid and Gr, Re, Ec and  $\lambda$  are the Grashof, Reynold, Eckert, and Richardson numbers, respectively.

$$Gr = \frac{g\beta_f(T_w - T_\infty)a^3}{{v_f}^2}, \quad Re = \frac{u_\infty a}{v_f}, \quad Ec = \frac{{v_f}^2 Gr}{a^2 \left(C_p\right)_f \left(T_w - T_\infty\right)}, \quad \lambda = \frac{Gr}{Re^2}.$$

In order to solve Eq. (6) - (8), the following function are introduced:

$$\psi = x f(x, y), \quad \theta = \theta(x, y), \tag{10}$$

Where  $\psi$  is the stream function which defined as;

$$u = \frac{\partial \psi}{\partial y}$$
 and  $v = -\frac{\partial \psi}{\partial x}$ 

Substituting Eq. (10) into Eq. (6) - Eq. (8), the following partial differential equations (PDEs) are obtained the momentum and energy equations:

$$\frac{v_{hnf}}{v_f} \frac{\partial^3 f}{\partial y^3} + f \frac{\partial^2 f}{\partial y^2} - \left[ \frac{\partial f}{\partial y} \right]^2 + \frac{(\rho \beta)_{hnf}}{\rho_{hnf} \beta_f} \frac{\sin x}{x} (\lambda \theta + \cos x) = x \left[ \frac{\partial f}{\partial y} \frac{\partial^2 f}{\partial x \partial y} - \frac{\partial f}{\partial x} \frac{\partial^2 f}{\partial y^2} \right]$$
(11)

$$\frac{k_{hnf}(\rho c_p)_f}{k_f(\rho c_p)_{hnf}} \frac{1}{Pr} \frac{\partial^2 \theta}{\partial y^2} + f \frac{\partial \theta}{\partial y} = x \left[ \frac{\partial f}{\partial y} \frac{\partial \theta}{\partial x} - \frac{\partial f}{\partial x} \frac{\partial \theta}{\partial y} \right] - \left[ x E c \frac{v_{hnf}}{v_f} \frac{\rho_{hnf}(c_p)_f}{(\rho c_p)_{hnf}} \left( \frac{\partial^2 f}{\partial y^2} \right)^2 \right]$$
(12)

Where  $Pr = \frac{v_f(\rho cp)_f}{k_f}$  is the Prandtl number. Other quantities are detailed as follows:

$$i. \ \frac{v_{hnf}}{v_f} = \frac{1}{(1-\phi_1)^{2.5}(1-\phi_2)^{2.5}(1-\phi_2)[(1-\phi_1)+\phi_1(\rho_{s1}/\rho_f)] + \phi_2(\rho_{s2}/\rho_f)}$$

$$ii. \ \frac{(\rho\beta)_{hnf}}{\rho_{hnf}\beta_f} = \frac{(1-\phi_2)[(1-\phi_1)\rho_f + \phi_1(\rho\beta)_{s1}/\beta_f] + \phi_2(\rho\beta)_{s2}/\beta_f}{(1-\phi_2)[(1-\phi_1)\rho_f + \phi_1\rho_{s1}] + \phi_2\rho_{s2}}$$

$$iii. \ \frac{k_{hnf}(\rho C_p)_f}{k_f(\rho C_p)_{hnf}} = \frac{k_{hnf}/k_f}{(1-\phi_2)[(1-\phi_1) + \phi_1(\rho C_p)_{s1}/(\rho C_p)_f] + \phi_2(\rho C_p)_{s2}/(\rho C_p)_f}$$

$$iv. \ \frac{\rho_{hnf}(C_p)_f}{(\rho C_p)_{hnf}} = \frac{(1-\phi_2)[(1-\phi_1)\rho_f + \phi_1\rho_{s1}] + \phi_2\rho_{s2}}{(1-\phi_2)[(1-\phi_1)\rho_f + \phi_1(\rho C_p)_{s1}/(C_p)_f] + \phi_2(\rho C_p)_{s2}/(C_p)_f}$$

The boundary conditions Eq. (9) become:

$$f(x,0) = \frac{\partial f}{\partial y}(x,0) = 0, \quad \theta((x,0) = 1,$$
 (13)

$$\frac{\partial f}{\partial y}(x,\infty) \to \frac{\sin x}{x}, \quad \theta(x,\infty) \to 0$$

The physical quantities of interest are the local Nusselt number  $Nu_x$  and the skin friction coefficient  $C_f$ , which are given by:

$$Nu_{\chi} = \frac{aq_{w}}{k_{f}(T_{w} - T_{\infty})}, \quad C_{f} = \frac{\tau_{w}}{\rho_{f}u_{\infty}^{2}}$$

$$\tag{14}$$

The surface shear stress  $\tau_w$  and the surface heat flux  $q_w$  are given by

$$\tau_{w} = \mu_{hnf} \left[ \frac{\partial \overline{u}}{\partial \overline{y}} \right]_{\overline{y}=0}, \qquad q_{w} = -k_{hnf} \left[ \frac{\partial T}{\partial \overline{y}} \right]_{\overline{y}=0}, \tag{15}$$

with k being the thermal conductivity. Using Eq. (5) and Eq. (10) give

$$C_f R e_x^{1/2} = \frac{1}{(1 - \phi_1)^{2.5} (1 - \phi_2)^{2.5}} \left( x \frac{\partial^2 f}{\partial y^2} \right)_{\bar{y} = 0} \text{ and } N u_x R e_x^{1/2} = -\frac{k_{hnf}}{k_f} \left( \frac{\partial \theta}{\partial y} \right)_{\bar{y} = 0}$$
 (16)

Furthermore, the velocity profile and temperature distributions can be obtained from the following relations:

$$u = f'(x, y), \quad \theta = \theta(x, y) \tag{17}$$

# 3. Methodology

Using the Keller-box method, the partial differential equations (PDEs) Eq. (11) and Eq. (12) with boundary condition Eq. (13) are numerically solved. Keller-box method is an effective implicit technique for solving non-linear parabolic partial differential equations (PDEs). This study employs the use of a nonlinear equation to solve heat flow problems. Na [23], Cebeci and Cousteix [24], and recently Mohamed [25], each source offers detailed explanations of the research methods used. Keller-box method involves the following four steps. The first step involves the reduction of the Partial Differential Equations (PDEs) subjected to the boundary conditions to a first-order system. The central finite difference approach is employed, and The Newton's method is used to linearize the given function. The algebraic equations obtained are then transformed into a matrix-vector form, and lastly, the technique of block tridiagonal elimination is used to solve them. The algorithm is coded into MATLAB program to numerically compute. To obtain accurate numerical findings, noting that the thickness of the boundary layer satisfied  $y_{\infty}=7-10$  with step size  $\Delta y=0.02$ ,  $\Delta x=0.005$  are used.

## 4. Results

Eq. (11) and Eq. (12) represent momentum and energy respectively, based on a number of parameters, namely Prandtl number Pr, Richardson number  $\lambda$ , Eckert number Ec and the quantities of hybrid nanofluid. The comparison values of the reduce Nusselt number,  $Nu_xRe_x^{-1/2}$  on the surface body have been computed. It is found that a very strong correlation between the findings of three researchers. Table 1 shows the values of thermophysical properties of water and nanoparticles [25, 26]. Table 2 presents the comparison between the present results with the previously reported results by Merkin [27], Nazar [28] and Mohamed [29] for various value of the Richardson number  $\lambda$  when  $\phi_1 = \phi_2 = Ec = 0$  and Pr = 1. Also, it is discovered that they are in agreement. Table 3 and 4 present the values of reduce Nusselt number,  $Nu_xRe_x^{-1/2}$  and reduce of skin friction,  $C_fRe_x^{1/2}$  respectively for various values of circular cylinder x and Richardson number  $\lambda$  when Pr = 7 and Ec = 0.1.

**Table 1**Thermophysical properties of water and nanoparticles [24, 25]

Physical properties	Water (f)	$Al_2O_3\left(\phi_1\right)$	$Ag(\phi_2)$	$TiO_2$	Си
$ ho$ (kg/m $^{3}$ )	997	3970	10500	4250	8933
Cp (J/kg.K)	4179	765	235	686.2	385
K (W/m.K)	0.613	40	429	8.95	400

Table 2 Comparison value of  $Nu_xRe_x^{-1/2}$  with previous published results for different values of x and  $\lambda$  when  $\phi_1=\phi_2$ , Pr = 1, Ec = 0

$\psi_2$ , $II - I$ , $I$	7C - O										
		-1.0				0				1.0	
$x/\lambda$ Merkin	Nazar	Mohamed	Present	Merkin	Nazar	Mohamed	Present	Merkin	Nazar	Mohamed	Present
[27]	[28]	[29]		[27]	[28]	[29]		[27]	[28]	[29]	
0.0 0.5067	0.5080	0.5067	0.5067	0.5705	0.5710	0.5705	0.5705	0.6156	0.6160	0.6156	0.6156
0.2 0.5018	0.5022	0.5016	0.5015	0.5658	0.5658	0.5669	0.5668	0.6115	0.6125	0.6126	0.6125
0.4 0.4865	0.4862	0.4862	0.4859	0.5564	0.5564	0.5564	0.5562	0.6028	0.6031	0.6037	0.6036
0.6 0.4594	0.4584	0.4589	0.4584	0.5391	0.5391	0.5389	0.5387	0.5885	0.5880	0.5892	0.5889
0.8 0.4160	0.4140	0.4151	0.4144	0.5145	0.5145	0.5143	0.5139	0.5686	0.5673	0.5690	0.5687
1.0 0.3326	0.3259	0.3301	0.3280	0.4826	0.4808	0.4823	0.4817	0.5435	0.5414	0.5436	0.5432
1.2				0.4426	0.4406	0.4414	0.4414	0.5133	0.5105	0.5126	0.5126
1.4				0.3928	0.3909	0.3912	0.3912	0.4785	0.4750	0.4774	0.4774
1.6				0.3280	0.3262	0.3258	0.3257	0.4394	0.4354	0.4381	0.4380
1.8				0.2114	0.2049	0.2043	0.2039	0.3967	0.3924	0.3951	0.3950
2.0								0.3509	0.3465	0.3500	0.3491
2.2								0.3029	0.3002	0.3013	0.3012
2.4								0.2540	0.2515	0.2526	0.2524
2.6								0.2061	0.2040	0.2051	0.2048
2.8								0.1634	0.1636	0.1632	0.1628
3.0								0.1354	0.1397	0.1371	0.1358
π								0.1306	0.1380	0.1327	0.1317

**Table 3** Values of  $Nu_xRe_x^{-1/2}$  with different values of x and  $\lambda$  when Pr = 7 and Ec = 0.1

	alues of $Nu_{\chi}Ne_{\chi}$	with an	ierent value.	o oi x aiiu x	WITCH I I - 7	and LC = 0.1
X/	/λ -1.0	-0.5	0.0	0.5	1.0	2.0
0.	0 1.0974	1.1406	1.1786	1.2126	1.2436	1.2985
0.	2 1.0808	1.1237	1.1609	1.194	1.2238	1.2761
0.	4 1.0337	1.0761	1.1115	1.1418	1.1684	1.2131
0.	6 0.9610	1.0038	1.0368	1.0633	1.0850	1.1178
0.	8 0.8679	0.9147	0.9463	0.9686	0.9844	1.0025
1.	0 0.7536	0.8152	0.8489	0.8682	0.8785	0.8807
1.	2 0.5849	0.7057	0.7506	0.7708	0.7772	0.7650
1.	4	0.5672	0.6511	0.6803	0.6869	0.6644
1.	6		0.5375	0.5945	0.6087	0.5836
1.	8		0.3193	0.5034	0.5384	0.5217
2.	0			0.3790	0.4674	0.4743
2.	2				0.3833	0.4341
2.	4				0.2673	0.3944
2.	6					0.3501
2.	8					0.2981
3.	0					0.2342
π						0.1676

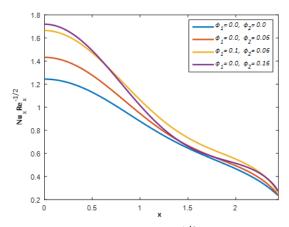
**Table 4** Values of  $C_f Re_x^{1/2}$  with different values of x and  $\lambda$  when Pr = 7 and Ec = 0.1

Values of $C_f Re_x$ with different values of x and $\lambda$ when $Pr = I$ and $Ec = 0.1$						
x/λ	-1.0	-0.5	0.0	0.5	1.0	2.0
0.0	0	0	0	0	0	0
0.2	0.168	0.2064	0.2426	0.2770	0.3100	0.3728
0.4	0.3117	0.3895	0.4623	0.5314	0.5976	0.7231
0.6	0.4082	0.5278	0.6384	0.7428	0.8423	1.0305
8.0	0.4372	0.6034	0.7539	0.8943	1.0274	1.2776
1.0	0.3783	0.6023	0.7965	0.9742	1.1409	1.4519
1.2	0.1901	0.5132	0.7593	0.9766	1.1769	1.5464
1.4		0.3174	0.6402	0.9018	1.1362	1.5604
1.6			0.4372	0.7562	1.0257	1.4996
1.8			0.1008	0.5500	0.8585	1.3754
2.0				0.2900	0.6525	1.2041
2.2					0.4288	1.0045
2.4					0.2103	0.7959
2.6						0.5947
2.8						0.4100
3.0						0.2369
π						0.1043

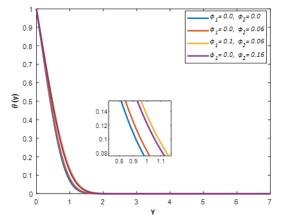
Figure 2 illustrates the variation of volume fraction of hybrid nanofluids against reduce heat transfer along the body circular cylinder. It was observed that the effect of variation in  $\phi_1$  and  $\phi_2$  are more significantly at the stagnation point (x=0). It is claimed that the reduce of heat transfer decreasing along a cylinder surface. The  $Al_2O_3-Ag/$ water ( $\phi_1=0.1$ ,  $\phi_2=0.06$ ) hybrid nanofluid score highest values in reduce of heat transfer compared to water-based fluid and Ag/water ( $\phi_1=0.0$ ,  $\phi_2=0.06$ ) nanofluid. These results are comparable with high-cost Ag/water ( $\phi_1=0.0$ ,  $\phi_2=0.16$ ) nanofluid. The ability of the hybrid nanofluid, which combines metal and low-cost oxide nanoparticles, to transport heat to surfaces is demonstrated to be on par with that of premium metal nanofluid.

Figure 3 show that values of skin friction is typical at the stagnation region (x = 0). At this point, the nanoparticles have almost no effect on friction. As fluid flows through the cylinder body, the skin friction  $C_f Re_x^{1/2}$  increases when 6 % vol. of Silver Ag nano material is added up into water-based fluid with the adding 10 % vol. of Alumina  $Al_2O_3$  nano oxide to form the  $Al_2O_3$ -Ag/water ( $\phi_1$  = 0.1,  $\phi_2$  = 0.06) hybrid nanofluid. From Figure 3, it was found that values of skin friction is unique and then increase again without adding nano oxide ( $\phi_1$  = 0.0). The highest values of skin friction with adding 6 % and 16 % nano material to form the Ag/water ( $\phi_1$  = 0.0,  $\phi_2$  = 0.06) and ( $\phi_1$  = 0.0,  $\phi_2$  = 0.16) nanofluid respectively. Considering  $Al_2O_3$ -Ag/water hybrid nanofluid has a concentration of nano oxide 10% and nano metal 6%, the friction is less than 16% Ag/water nanofluid. As a result, it is safer for the surface and increases its usefulness. Generally, the highest  $C_f Re_x^{1/2}$  occurs when the fluid passes through the surface body of cylinder x between 1.0-1.5 radian.

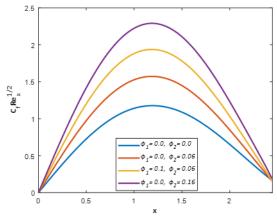
The temperature profiles  $\theta(y)$  and velocity profiles  $f'(\eta)$  at stagnation region (x=0) for various values of  $\phi_1$  and  $\phi_2$  are showed in Figure 4 and Figure 5, respectively. It is obvious that the presence of more nanoparticles has raised the thickness of the thermal boundary layer and decreasing the thickness of the velocity boundary layer. The addition of nanoparticles in hybrid nanofluid has increased the thermal conductivity of fluid, raising thermal diffusivity and increasing thermal boundary layer thickness. This is realistic especially for nano oxide has higher specific heat than nano material to store energy. Adding up nano oxide to Ag/water to form the hybrid nanofluid will increase the thermal boundary layer thickness. It means, increasing of thermal boundary layer thickness will increase heat transfer.



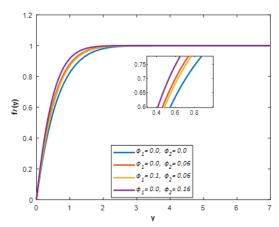
**Fig. 2.** Variation of  $Nu_xRe_x^{-1/2}$  against x for various of volume fraction  $\phi_1$  and  $\phi_2$  When Pr=7,  $\lambda=1$  and Ec=0.1



**Fig. 4.** Temperature profile  $\theta(y)$  against y for various of volume fraction  $\phi_{-}(1)$  and  $\phi_{-}(2)$  When Pr=7,  $\lambda = 1$  and Ec=0.1



**Fig. 3.** Variation of  $C_f Re_x^{1/2}$  against x for various of volume fraction  $\phi_1$  and  $\phi_2$  When Pr=7,  $\lambda=1$  and Ec=0.1 When Pr=7,  $\lambda=1$  and Ec=0.1



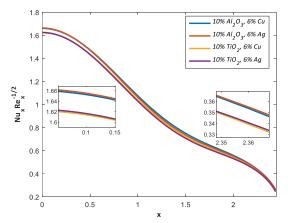
**Fig. 5.** Velocity profile f'(y) against y for various of volume fraction  $\phi_1$  and  $\phi_2$  When Pr=7,  $\lambda=1$  and Ec=0.1

Next, Figures 6 and 7 show the variation reduce Nusselt number and reduce skin friction for various values of concentration of hybrid nanofluid when Pr=7,  $\lambda=1$  and Ec=0.1. Respectively. The flow and heat transfer performance of nano oxide  $Al_2O_3$  and  $TiO_2$  with Ag/water and Cu/water to form hybrid nanofluid. From the stagnation point, x=0 to 0.7 radian, Ag/water transfer energy extremely well, next from 0.7 to 2.1 radian, Cu/Water dominates in transferring energy, lastly from 2.1 to 2.4 radian, Ag/water carries energy to completion (see Figure 6). This is due to the fact that the thermal conductivity of Ag and  $Al_2O_3$  is larger than that of Cu and  $TiO_2$ . According to James [30], The thermal conductivity is defined as ability of the substance to transfer energy. Figure 7, In general, the maximum  $C_fRe_x^{1/2}$  is obtained when the fluid flows through the surface body of cylinder x between 1.0 and 1.5. In comparison to Ag/water nanofluid and  $TiO_2$ nano oxide, it was discovered that Cu/water nanofluid and  $Al_2O_3$  nano oxide has a higher efficiency in reducing skin friction. This is due to the lower density of Cu and  $Al_2O_3$  compared to Ag and  $TiO_2$ .

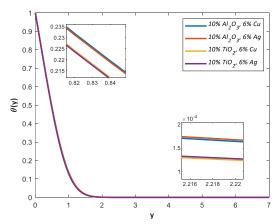
The temperature  $\theta(y)$  and velocity profiles  $f'(\eta)$  at stagnation region (x=0) for various concentration of hybrid nanofluid are showed in Figure 8 and Figure 9, respectively. In comparison to the hybrid nanofluids of  $Al_2O_3$ -Cu/water,  $TiO_2$ -Ag/water and and  $TiO_2$ -Cu/water respectively,

the Figure 8 demonstrates that the  $Al_2O_3$ -Ag/water hybrid nano fluid has an effective heat transfer beginning at 21.4% (1.5 radian) of the maximal boundary layer thickness on the surface. From Figure 9 show, in overall, the peak thickness of the velocity boundary layer occurs between 15% and 17% of the maximum thickness of the velocity boundary layer. The hybrid nanofluid of  $TiO_2$ -Ag/water has a highest of velocity boundary layer thickness compare hybrid nanofluid of  $Al_2O_3$ -Ag/water,  $Al_2O_3$ -Cu/water and  $TiO_2$ -Cu/water respectively. The effect of the density of a particle also impacts the thickness of the velocity boundary layer.

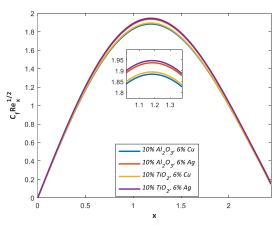
Lastly, Figure 10 and Figure 11 illustrate the Reduce Nusselt number and skin friction against x for various values of Eckert number. Both the figures agreed that the viscous dissipation effect Ec are negligible at stagnation region (x = 0). The viscous dissipation effect is more apparent near the middle of cylinder body. Increasing viscous dissipation of the Eckert number parameters decrease the temperature distribution with no significant change in the fluid velocity [31]. The highest Ec value indicates the low temperature distribution on the surface of the cylinder body and the Ec value also show no significant in reducing skin friction, both of the statements can be seen in Figure 10 and Figure 11 respectively.



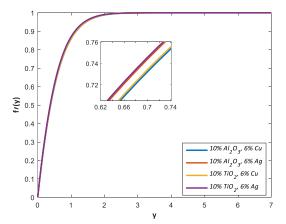
**Fig. 6.** Variation of  $Nu_xRe_x^{-1/2}$  against x for various concentration of hybrid nanofluid When Pr=7,  $\lambda=1$  and Ec=0.1



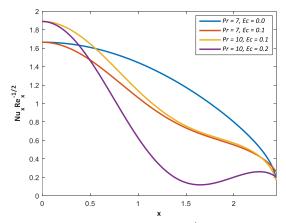
**Fig. 8.** Variation of temperature profile against boundary layer thickness for various concentration of hybrid nanofluid When Pr=7,  $\lambda=1$  and Ec=0.1



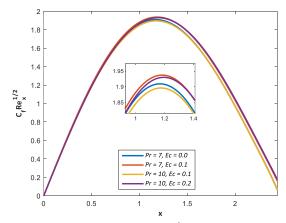
**Fig. 7.** Variation of  $C_f Re_x^{1/2}$  against x for various concentration of hybrid nanofluid When Pr=7,  $\lambda=1$  and Ec=0.1



**Fig. 9.** Variation of velocity profile against boundary layer thickness for various concentration of hybrid nanofluid When Pr=7,  $\lambda=1$  and Ec=0.1



**Fig. 10.** Variation of  $Nu_xRe_x^{-1/2}$  against x for various values of Ec, when  $\lambda = 1$ 



**Fig. 11.** Variation of  $C_f Re_x^{1/2}$  against x for various values of Ec, when  $\lambda = 1$ 

## 5. Conclusions

In this paper, the mixed convection boundary layer flow over a horizontal circular cylinder in hybrid nanofluid was investigated using the Keller-box method. The effects of the Richardson number  $\lambda$ , the Eckert number Ec, and the volume fractions of oxide nanoparticles and metal nanoparticles in hybrid nanofluid are documented. As a conclusion, it is found that the increase of volume fraction of nano material in nanofluid has increased the value of skin friction coefficient. In hybrid nanofluids, the low density of nano oxides such as alumina contributes to decrease friction between the fluid and body surface. The selection of nano oxide can be considered a reference for radiator coolant in automotive applications. The combination of nanoparticles in the form of  $Al_2O_3$ -Ag/water hybrid nanofluid may reduce skin friction phenomena while maintaining heat transfer characteristics comparable to Ag/water nanofluid, as determined by numerical investigation.

Following that, it is observed that skin friction encountered increased amounts of friction in the middle of the cylinder's surface, while the Nusselt number demonstrates a decreasing variation along the circular cylinder. An increase in the Eckert number results in a related rise in friction, at x=1.0 to 1.5 on the surface of the cylinder, the greatest friction is evidently present. Although the skin friction effect is intended to have the opposite effect, an increase in Eckert number results in a decrease in Nusselt number.

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